2	Burning characteristics of single particles of coal and wood
3	mixtures for co-firing in an upward-flowing hot gas stream
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36	Abstract
37	This study presents the comparative burning behaviours of single solid particles of coal and

³⁷ This study presents the comparative burning behaviours of single solid particles of coal and ³⁸ biomass mixtures for co-firing. In this experimental investigation, a direct observation ³⁹ approach was used to investigate the ignition, flame characteristics and combustion times by ⁴⁰ means of high-speed photography at 7,000 frames per second. Single particles were entrained ⁴¹ into a hot gas stream at 1,340 K and a rapid heating rate at 10^4 - 10^5 K/s. The apparent volatile ⁴² flames from the prepared particle size groups were observed within 20-50 milliseconds. To ⁴³ assess the effect of oxygen concentration, particles were burned for their flame characteristics ⁴⁴ in a range of 10%–40% O₂. The test particles were sieved into three size groups (215–255 µm,

45	255–300 μm and 300–350 $\mu m)$ to assess the effect of particle size. Special particles for the
46	co-firing effect were collected individually from two types of mixed pellet: 20:80 and 50:50
47	coal/wood. Pure sub-bituminous coal and wood particles were also prepared in order to
48	compare their combustion behaviours. In the experimental setup with a cross-injection
49	configuration, sequential combustion processes were effectively and clearly described in
50	terms of particle displacement with time. The experimental results showed distinguishable
51	flame characteristics from single particles of coal, 50:50 coal/wood, 20:80 coal/wood and
52	wood, including soot flame size and intensity. The impact of high coal-blending ratio caused
53	an increase in the flame size and intensity and the ignition time was close to that of pure coal
54	particles. Quantitative measurements of combustion events on co-firing particles were also
55	discussed in relation to significant impacts of the particle size and the oxygen concentration.
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66 **1. INTRODUCTION**

Co-firing is a promising technique with several benefits, the most important of which are 67 68 environmental ones. Simultaneous burning of coal and biomass emits low NET carbon dioxide [1, 2] and the sulphur oxide (SO_x) and nitrogen oxide (NO_x) emissions from this 69 70 combustion are generally lower, compared with the coal-fired combustion [3-6]. The economic benefits of this type of combustion are fuel price stabilisation, fuel flexibility and 71 72 low-risk costs of retrofitting coal-fired plants with fuel-blending control systems for biomass 73 particles [7, 8]. The main challenges with co-firing come from the characteristics of biomass particles in relation to chemical composition, fibrous shape and relatively large particles (~3 74 75 millimetres for biomass, ~150 µm for coal in p.f. boiler) [9, 10] with relatively low energy 76 density [11, 12]. Their irregular shape and wide particle size range during combustion play a 77 significant role in burning behaviours such as ignition, volatile flame and char combustion 78 [13]. The significant differences between biomass and coal are the wide disparity in volatility 79 and the fixed carbon content of different types of biomass. The volatile matter content of biomass is greater than that of coal and the fixed carbon content is less, which contributes to 80 their different burning behaviours regarding volatiles and char combustions [14–16]. From a 81 previous study [17], pulverised biomass particles are known to exhibit flame characteristics 82 83 that are associated with a small flame and low intensity during homogeneous combustion. 84 Besides, the volatile matter in biomass is released earlier than that in coal because of the different ignition temperatures of their organic compositions [18–20]. Consequently, the 85 burning behaviour in co-firing is attributed mainly to the biomass type and its mixtures. 86

Soot loading in volatile flames plays an important role in generating radiation heat transfer, because of soot luminosity, as illustrated by the equation: $q_{radiation} = \epsilon 4\sigma T^4 \Delta t$, where $\epsilon = 1$ -(1+kf_vLT/c₂)⁻⁴ [21, 22]. There have been studies on average soot volume fraction from 90 different types of coal [15]. It is believed that biomass has less flame luminosity, where gas is 91 the major phase in the flame. Atiku et al. [24] measured soot particles, which consist of 92 elemental carbon (EC) and organic carbon (OC), between biomass and coal. During flaming 93 combustion, eight times as much elemental carbon was collected from burning coal. In the 94 experiment, the apparent volatile flame would be discussed for comparative soot intensity 95 between co-firing particles and pure solid particles.

96 An in-depth understanding of the thermal decomposition and combustion in co-firing would certainly be beneficial. Several studies [2, 7, 18-20, 24-28] have reviewed technical 97 98 considerations and conducted biomass combustion with coal in extended experimental investigations. In thermogravimetric analysis (TGA), the devolatilisation of biomass-coal has 99 been investigated extensively at low heating rates, with focus on the ignition characteristics 100 101 [18], kinetic analysis [19] and primary reactions [25]. However, the experimental results of such studies have not been adequately explained in the context of practical combustion 102 103 behaviour. This is because the actual pulverised solid-fuel particles in power plants are generally burned rapidly and are accompanied by complicated reactions and events. Gani et 104 al. [26] investigated the co-combustion of coal with biomass in relation to ignition, NO_x 105 106 emission and ash formation in a drop-tube furnace at 800°C. They plotted the temperature profile along the furnace during biomass, coal and co-combustion and showed that co-107 combustion yields the highest temperature of the three. Lu et al. [27] reported that adding 108 109 biomass can prolong ignition in co-firing because of the large particles and high moisture content of the biomass. They also investigated flame characteristics such as brightness, 110 temperature and ignition in industrial-scale coal combustion. However, there were 111 uncertainties such as the loading of particles, their sizes and environmental conditions; these 112 limited the extent to which they were able to explain the flame parameters. Therefore, for a 113

114 comparative analysis to clarify the combustion behaviour of solid-fuel particles with clearly 115 defined physical properties under well-controlled conditions, it is necessary to burn one 116 particle at a time.

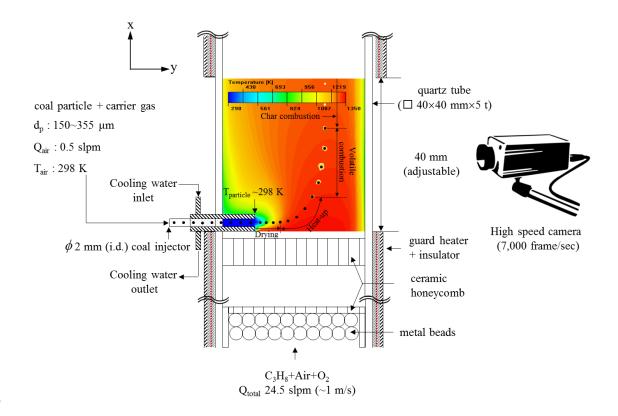
In previous studies [17, 29], we investigated the combustion of single coal and biomass 117 particles in a lab-scale entrained-flow reactor. The experiments were carried out under rapid 118 119 heating rates and a high gas temperature in order to describe the particle combustion process 120 and investigate the flame characteristics. Khatami, Levendis et al. [30-32] also observed the 121 behaviour of single particles of individual coal and biomass in a drop-tube furnace. However, 122 the observations of flame structure from these experiments are neither fully understood nor explained in relation to co-firing. Different physical and chemical events can occur in co-123 firing because of a mixture of solid fuel particles. Unfortunately, no experimental research 124 has been reported on single particles of pulverised coal and biomass mixtures. 125

This paper reports an investigation of burning single particles, which were produced from a 126 127 mixture of pulverised coal and biomass, using direct observation. Sequential combustion 128 processes such as heat-up, devolatilisation and char combustion are described in detail. The 129 effects of having single particles with different blending ratios, sizes and oxygen concentrations are also discussed. Given the statistical uncertainties related to single particles 130 131 of homogenous fuel mixtures (20:80 and 50:50 mixtures of coal and wood), there was a clear need for particle definition from sub-experiments such as scanning electron microscopy 132 (SEM), fuel properties and TGA, prior to the main experiment. The experimental results are 133 compared to discuss their ignition and combustion time and flame parameters with pure wood 134 and coal single particle burning behaviours through quantitative analysis. 135

136 2. METHODOLOGY

137 2.1 Experimental apparatus

The experimental combustion study of solid fuel particles was performed in a lab-scale 138 139 entrained-flow reactor as shown in Fig. 1 with optical access to the burning single particles. The square-shaped quartz tube is 45 mm in length and width and 800 mm in height, and a 140 honeycomb burner is located at the bottom for an upward flow of post-combustion gas. An 141 electrical guard heater shields the quartz to minimise the temperature drop of the post-142 143 combustion gas except across the optical access section. Particles are injected at ambient 144 temperature (298 K) via a water-cooled injector made of stainless steel double tubes: an outer 145 tube for cooling water and an inner tube for the carrier gas and particles. This injector is positioned above a porous flow straightener in the cross-flow configuration in which single 146 particles are entrained from the horizontal cold carrier gas into the vertical hot gas stream. 147 148 The injector is also placed 20 mm from the interior wall to minimise temperature and flow differences between the hot gas stream and the wall. To support the temperature environment, 149 150 the temperature gradient in the optical access section is shown in Fig. 1 by a simulation image (Fluent 13.0) that includes a prediction of particle displacement during burning. The 151 carrier-gas flow rate is 0.5 slpm in all experiments and the particles are injected at 25-30 152 153 particles/min by a micro-syringe injector.



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Fig. 1. Schematic diagram of the lab-scale entrained-flow reactor for a few hundred micronsized single solid particles. Also shown are the temperature fields of the hot gas stream (red) and cold carrier gas (blue), which explain the temperature gradient and the predicted sequential combustion processes of burning particle in the optical access section.

159 2.2 Environmental conditions

The variable-supply gas inputs of C_3H_8 , O_2 and air into the honeycomb are controlled for 10%–40% oxygen concentration, whereas the flow rate of these gases is maintained identically at 24.5 slpm for the experiments. The following main flue gases are produced: N₂ (38.2%–67.9%), O₂ (10.4%–40.1%), H₂O (12.4%) and CO₂ (9.3%).

The gas temperatures were measured on the basis of radiation loss of the probe of an R-type thermocouple at 21 locations in x- and y-axis: 0, 7, 14, 21 and 50 mm from the injector along the y-axis and 2, 19, 20, 30 and 39 mm along the x-axis. To obtain a mean value, temperature 167 was averaged over 30 seconds. The variation in adiabatic flame temperatures from the post-168 combustion was approximately 140 K from 10 % to 40 % oxygen concentration. The mixing 169 zone of the leftward-flowing cold carrier gas and the upward-flowing hot main gas stream is 170 shown in Fig. 2. As particles pass through this zone, their temperature increases up to the devolatilisation or ignition temperature. The highly interacting flow zone at 0 mm along the 171 172 y-axis is also sufficient to affect particle motion and combustion. Khatami et al. [31] discussed the change in combustion behaviour of a particle between quiescent gas conditions 173 174 (no flow) and active gas flow. Therefore, the environment with the mixing zone and cross injection configuration enables an exclusive description of sequential combustion processes 175 such as drying, heat-up and volatile combustion regimes that accompany particle motion. 176

The velocities of post-combustion gas and cold carrier gas in a cross-jet apparatus have been discussed by Lee et al. [33]. The particle velocities during the release of volatile matter vary according to these physical quantities, so all experiments were performed under identical vertical and horizontal velocities: ~1 m/s for the post-combustion gas and 2.5 m/s for the cold carrier gas.

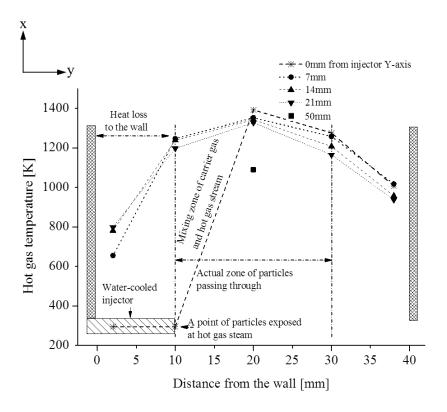
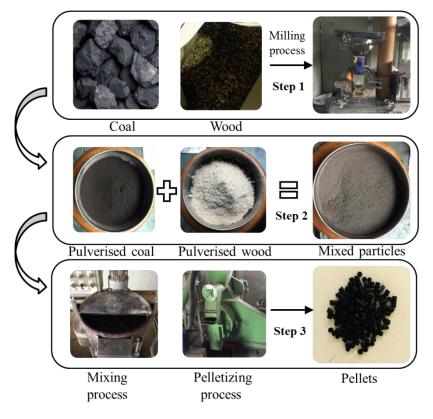


Fig. 2. Experimental measurements of surrounding gas temperatures in the optical access
section. This shows the actual zonal temperatures of particles passing through and the
predicted initial particle temperature before heat-up.

186 **2.3 Particle properties**

The coal used in this study was a sub-bituminous type mined by Adaro Energy in Indonesia; 187 the biomass was pine wood logged in Gyeongsang Province, South Korea. These materials 188 were used for making a few-hundred-micron-sized single particles with a coal-to-wood fuel 189 ratio of either 50:50 or 20:80. To produce these special particles, we first created pellets in a 190 pellet factory through a sequence of pulverising and sieving the raw coal and wood (step 1), 191 192 mixing the resulting powders (step 2) and compressing the mixture into pellets (step 3), as shown in Fig. 3. In step 1, particles are milled separately by species and sieved to under 193 194 100 µm; anything larger can spoil a homogeneous mixture. Step 2 is an initial fuel mixing process in which the same volume (500 mL) of coal and wood or 200 mL and 800 mL, 195

respectively, are added to a container before being shaken and then sieved for 20 min. The final step is the pelletizing process, which involves another mixing process with the addition of 2 kg of water per 1 m² of particles. This water content plays an important role in the physical bonding of different solid fuel particles as an adhesive source; this particular level of water loading was determined through repeated tests of pellet stability/integrity.



Step 1: Milling process of coal & woodStep 2: Mixing process of pulverised coal & woodStep 3: Pelletizing process of mixed particles

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Fig. 3. Preparation of pellets with 50:50 or 20:80 coal/wood mixtures. This figure shows how

the pellets were made with fuel blending for co-firing, prior to being pulverised into a few-

204 hundred-micron-sized particles.

Two types of pulverised co-firing particles were obtained from several separation methods that have been described in detail previously [17]. Some of the collected particles inevitably 207 have irregular shapes with high aspect ratio because of the fibrous wood particle structure, but these can be separated using an inclined plane. The particles were separated into size 208 209 groups (215–255 µm, 255–300 µm and 300–355 µm) using seven testing sieves with different mesh diameters: 215, 255, 300 and 355 µm. Repeated tests with uncoated paper on an 210 inclined plane allowed the particles to be sorted by shape, e.g., flat, cylindrical, or spherical. 211 In the experiment, only the spherical-like shape is used because the cross-jet injection is 212 limited in burning irregular particle shapes. Flat-like and cylindrical-like particles lead to 213 very random trajectories with non-uniform particle motion in the reactor. 214

215 Four particles were subjected to chemical composition analysis on an as-received basis, as shown in Table 1. These analysis data were obtained using the thermogravimetric analyser 216 (TGA-701) at the Energy and Environment Research Centre of the Korea Advanced Institute 217 of Science and Technology (KAIST). The changes in volatile matter and fixed carbon mass 218 fraction for the 50:50 blended particle were approximately +15% and -16.6%, respectively, 219 220 compared with coal particles, whereas for the 20:80 blended particle they are +26.7% and -27.7%, respectively. The blended particles have higher moisture content because of the 221 water added during pelletising. However, the moisture differences between the four particles 222 223 after the pulverising process are too small to noticeably affect their combustion behaviours.

The bulk densities of the four particles were measured using a simplified bulk density technique that involved filling a bottle with pulverised particles. This method allows the density of each packed-bed particle to be calculated based on $\rho_p = (m_{b,p}-m_{b,e})/V_{b,w}$, where ρ_p is the particle density, $m_{b,p}$ is the mass of the bottle filled with particles, $m_{b,e}$ is the mass of the empty bottle and $V_{b,w}$ is the volume of the bottle determined from water filling. Using coal for the reference bulk density, the relative bulk densities of 50:50 coal/wood, 20:80 coal/wood and wood are 570 kg/m³, 430 kg/m³ and 340 kg/m³, respectively. From measuring these particle densities, the energy densities were calculated as 19.581 GJ/m^3 for coal, 12.490 GJ/m³ for 50:50 coal/wood, 8.530 GJ/m³ for 20:80 coal/wood and 6.215 GJ/m³ for wood.

The difference in chemical composition between coal and wood particles is shown in Fig. 4 in a Van Krevelen diagram that indicates the relationship between the hydrogen/carbon (H/C) and oxygen/carbon (O/C) ratios. Coal particles have the lowest H/C (0.73) and O/C (0.28) ratios, whereas wood particles have the highest H/C (1.34) and O/C (0.89) ratios. The latter contain highly volatile matter that is high in oxygen and low in fixed carbon. The presence of high oxygen content plays a significant role in the reactivity of the pyrolysis, and the tar yield is proportional to the H/C ratio [34].

241	Table 1.	Chemical	compositions	of four samples

Sample	Proximate analysis (wt. % ar) ¹			Ultimate analysis (wt. % daf) ²					LHV ^{3&1} (MJ/kg)	Particle bulk density	Approx. energy	
	V.M	F.C	Ash	М	с	н	0	Ν	s		(kg/m ³)	density (GJ/m ³)
Coal	42.0	46.1	1.5	10.4	64.4	4.7	18.1	0.9	0.1	25.43	770	19.581
Coal-Wood (50:50 %)	57.0	29.5	1.3	13.0	58.6	6.2	28.1	0.6	0.1	20.82	570	12.457
Coal-Wood (20:80 %)	68.7	18.4	0.6	11.7	52.3	6.4	35.4	0.4	0.1	17.14	430	8.530
Wood	81.5	9.7	0.1	8.5	48.5	6.5	43.2	0.2	0.1	25.43	340	6.215

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¹as received ²dry, ash free ³lower heating value

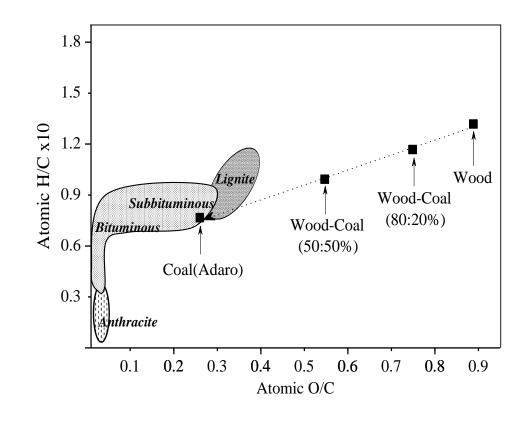


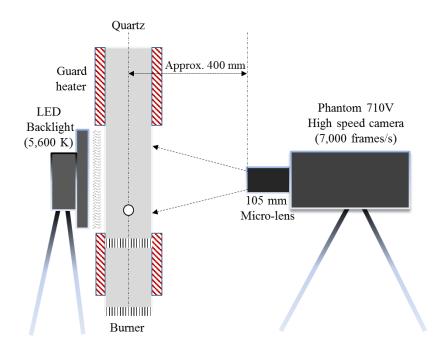
Fig. 4. Van Krevelen diagram for the four different solid fuel particles.

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247 **2.4 Direct observation of burning particles**

A high-speed camera (Phantom V710) was used for direct observation of the burning 248 particles. It can record at 7,000 frames per second and is fitted with a micro-lens (Nikon 249 Micro-NIKKOR 105 mm f/2.8) as shown in Fig. 5. The magnification of the microscope lens 250 251 was determined through repeated tests with 30-mm and 200-mm micro-lenses to record clear 252 images of the soot flame in the maximum range of combustion regimes. The complementary metal oxide semiconductor (CMOS) image sensor in the camera was $25.6 \text{ mm} \times 16.0 \text{ mm}$ 253 and a resolution of $1,280 \times 800$ pixels was selected to provide a quantitative analysis of the 254 255 burning particles. Lee et al. [29] introduced the calibration method with minimal observation 256 errors wherein the pixel size was calibrated using circle and line-scale reticle shapes, fitted inside the field of view of the camera. Without a backlight, the flame intensity on a particle's 257

surface was too bright to allow char particles to be captured simultaneously. The flame intensity varied according to the type of solid particle; for example, a coal particle burns with almost twice the intensity of a biomass particle [17]. Therefore, determining the backlight intensity played an important role in comparing the flame structures of the four particles. A controllable-brightness, light-emitting diode (LED) backlight with uniform luminescence and high intensity was adapted identically at 5,600 K for all experiments and was located at 50 mm from the quartz window.



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Fig. 5. Configuration of the high-speed camera and LED backlight. This enables the observation of particle ignition and volatile and char combustion on the few-hundred-micronsized surfaces.

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270 **2.5 Physical formation of the four particles from SEM images**

To explain the particle structures of the fuel mixtures, pulverised particles were examined using a NOVA 230 SEM. The resulting images shown in Fig. 6 were taken under the following conditions: accelerating voltage 10.0 kV, spot size 3.0, magnification 650–800×. The pure wood and coal particles were compared with the two co-firing particles. First, the raw wood particle had a remarkably fibrous structure with a high aspect ratio, and the external physical structure of the coal particle was an irregular blocky shape with angular transitions. These appreciably different shapes can play an important role in particle motion, temperature gradient and ignition [33, 35].

279

They may also exert a strong influence on differences in combustion behaviour, such as 280 flame structure, release of volatile matter and combustion mode. The blended particles were 281 282 physical mixtures of coal and wood fragments, and the small coal particles could appear 283 inside or on the particles. The fuel blending ratios (50:50 or 20:80) were fairly constant according to the volume fractions of coal particles used in the pelletising process. However, it 284 is difficult to guarantee a homogeneous mixture of pulverised particles. Repeated tests 285 showed remarkable differences in the apparent flame structures of each particle group but 286 similar combustion behaviour in identical particle groups. The standard deviation was less 287 than 10% in all experiments. We analysed the thermal decomposition of the four particles to 288 support whether the particle mixtures were homogeneous or not, as discussed later in section 289 290 3.

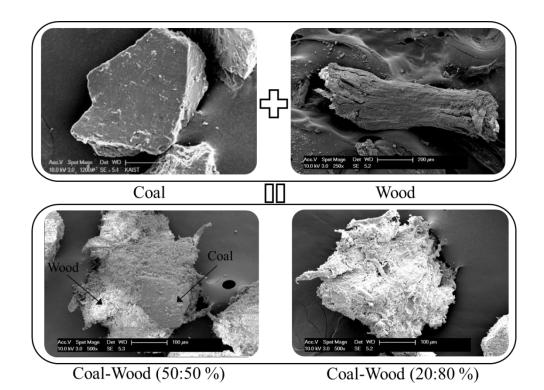


Fig. 6. SEM images of coal, wood, 50:50 coal/wood and 20:80 coal/wood particles.

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294 **2.6 Characterisation of volatile flame on pulverised coal and raw wood particles**

295 Flames comprise pyrolysis and luminous regions in which a gas phase and a solid phase (as submicron soot particles) are released by the chemical reactions. The luminous region is 296 occupied in a flame and the high radiative energy is generated mostly from the soot particle 297 298 fraction. The fixed carbon content plays an important role in the formation of an elongated 299 soot cloud, because this content is equivalent to elemental carbon, which is one of the 300 elements that produce soot particles [36]. Here we discuss the physical appearance of flames due to coal and wood particles prior to the investigation of single particles with different fuel 301 302 mixtures. As shown in Fig. 7, the different size and shape in volatile flames for both subbituminous coal and wood particles at 1,340 K were observed under 10-40 % O2. The 303 diffusion flame of coal was found to be several times larger than the wood flame under all 304

305 oxygen concentrations. Increasing the coal particle size under low O₂ concentration enhanced the elongated flame. This result is consistent with previous research [29, 37]. The attached 306 307 flame is formed where the rate of volatile release is balanced by the rate of oxidation, but this fast oxidation rate decreases the radius of the volatile flame and enhances the detached flame 308 depending on biomass species, particle size and particle heating rates. Especially, biomass 309 particles have thin volatile flames under high oxygen concentration because of their turbulent 310 dispersion of gas-phase, which is the major species in a flame. Coal particles under low O₂ 311 312 concentration have an elongated flame with a soot tail. This tail is formed by the effect of thermophoresis (force and velocity) between the solid phase and surrounding gas and low 313 availability of oxygen [38–40]. The differences in this soot tail are less pronounced at higher 314 315 oxygen concentration [41].

The flame characteristics of pure coal and wood particles offer a meaningful comparison for 316 the flame parameters of the two co-firing particles with different fuel-blending ratios. In the 317 experimental results, the intensities of volatile flames for the four particles were analysed by 318 image processing, which converts a flame image into a grayscale image. In the process, each 319 pixel in the image has different values in a range of 0 to 255, and numerical background 320 321 values, which were more or less than 65, are extracted from the images. After that, the intensity of the flame was divided by the maximum value of grayscale (255). The intensity-322 weighted values in the flame boundaries were fluctuating, which means that it might be 323 324 considered a flame threshold. To minimise their fluctuation and verify the minimised variation of the flame size, we have carried out the sensitivity analysis with varying back-325 lighting intensity until the rate of change in the flame size was minimised. 326

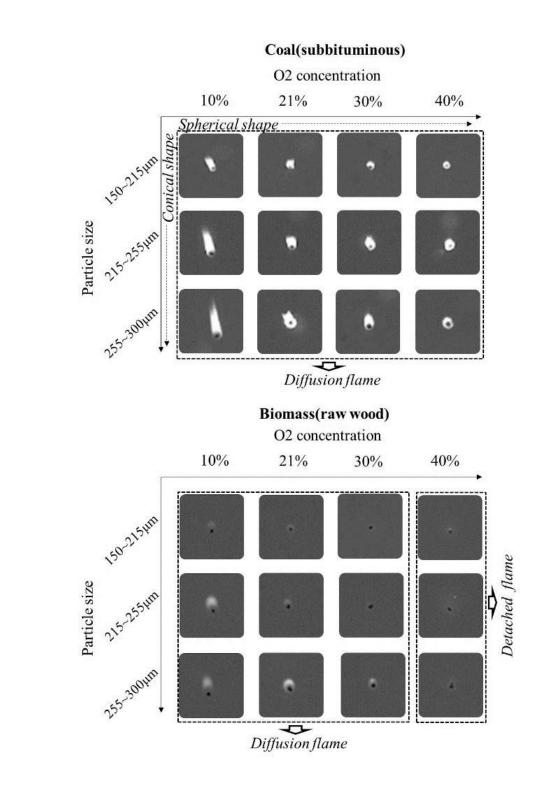


Fig. 7. Typical flame structures of pure coal and wood in relation to shape and size under 10– 40% O_2 and at 1,340 K. This illustrates the physical changes due to particle size and O_2 concentration, and explains the corresponding volatile combustion modes of attached,

334 diffusion and detached flames.

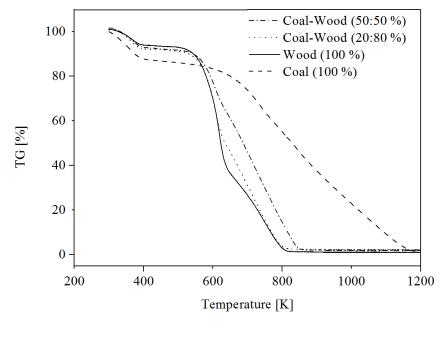
335 3. RESULTS AND DISCUSSION

336 **3.1 Sequential combustion processes under slow heating and thermal decomposition for**

337 different fuel-blending ratios

It is generally accepted that the impact of co-firing sub-bituminous coal (Adaro) with 338 339 biomass (pine wood) contributes to rapid devolatilisation, fast ignition and low emission of carbon dioxide because of the chemical and physical characteristics of wood. 340 Thermogravimetric analysis, TG/DTA 92-18, was used to investigate the characteristics of 341 342 co-firing particles with different fuel blending ratios compared to raw wood particles. Each sample was prepared at 8 ± 0.5 mg from pulverised particles in the range of 150-215 µm, and 343 the samples were heated at 20 K/min to 1,273 K under air conditions. The results explained 344 345 the thermal decomposition and how the mass of these samples reduces differently as a function of temperature over time. The main experiment was carried out under rapid heating, 346 347 but the TGA here was done under slow heating that is not representative of practical 348 conditions in an industrial furnace. In spite of this limitation, the different thermal decompositions of 20:80 and 50:50 coal/wood and wood particles can be discussed in relation 349 to the impact of co-firing combustion and by comparing the combustion behaviour with that 350 under rapid heating rate. Thermogravimetric (TG) plots and derivative thermogravimetric 351 (DTG) curves, which are obtained from the derivatives of the TG profiles, are shown in Fig. 352 8. The results show unequal TG and DTG curves, with the first and second peaks of mass 353 354 reduction followed by fluctuations between 50:50 coal/wood, 20:80 coal/wood and raw wood. In relation to the combustion processes, the first peak is due to the drying process that occurs 355 at 300-400 K, after which the particle is heated up with negligible mass reduction. 356 Sequentially, the volatile release starts at approximately 520 K and the peak reaches almost 357

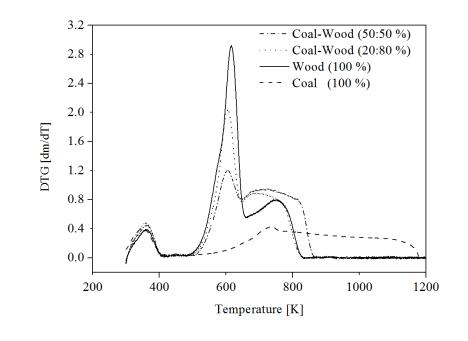
610 K because pure wood particles have three components: lignin, cellulose and 358 hemicellulose. A fluctuation then occurs because of lignin decomposition. Generally, 359 360 hemicellulose decomposes in a range of 493-588 K, cellulose does so at 588-673 K and lignin does so over a wide range of 433–1173 K [42]. Finally, the particle mass decreases at 361 constant mass loss rate, which is associated with char combustion. The second peak and 362 degree of fluctuation magnitude were obviously different between the three particles, which 363 is attributed to either the volume of wood particles or the volatile mass fraction in a particle. 364 Coal particles have a moderate peak and the char combustion regimes are extended because 365 of their high carbon content. Consequently, under slow heating, the highest peak for 366 devolatilisation occurs for the raw wood particle, and the co-firing particles also have higher 367 368 peaks at relatively low temperatures. However, the flame ignition and combustion behaviours under this slow heating rate does not correspond with those of a single burning particle under 369 rapid heating rate. This aspect is discussed in section 3.2. 370



371

372

(a)



(b)

373

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Fig. 8. Thermal decompositions of 50:50 coal/wood, 20:80 coal/wood and pure wood obtained from TGA at a heating rate of 20 K/min: (a) TG, (b) DTG. These graphs explain the distinction between mass reduction associated with homogeneous and heterogeneous combustion between the two prepared co-firing particles, coal particles and pulverised wood particles.

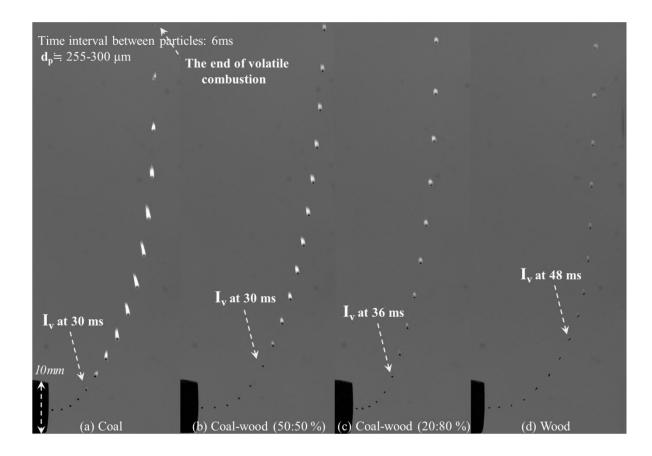
380

381 **3.2 Sequential combustion processes under rapid heating and development of volatile**

382 flames for different blending ratios

The sequential combustion processes of a pulverised solid-fuel particle are explained in the superimposed images in Fig. 9. Single particles of coal, 50:50 coal/wood, 20:80 coal/wood and wood in size ranges of 215–255 µm and 300–355 µm were burned under 10% oxygen concentration and at 1,340 K. Particles are heated rapidly by the drying process after particle injection at 298 K. The release of volatiles then starts within a few milliseconds, followed by volatile combustion. Char combustion starts after extinction of the volatile flame, but this

389 regime is not shown because of limited optical access. Under rapid heating rates, volatile combustion presents changes in the flame size, following volatile ignition. Comparative 390 391 flame size and intensity with sequential combustion processes can be explained as a function of time. In contrast, TGA, which is operated under slow heating rates, shows the fluctuated 392 profile with a peak for the regime of volatile combustion, providing a range of particle 393 temperature for volatile and char combustion and a different rate of devolatilisation between 394 the four particles. The similar tendency of volatile combustion was shown in the results, but 395 396 there are more complex physical and chemical events in a flame around a particle under rapid heating. Generally, high soot formation in a flame is associated with tar reacting to become 397 submicron-sized soot particles at high temperatures. This affects the volatile combustion 398 399 associated with highly luminous flames [43, 44]. From this result, the four burning particles differed in apparent homogeneous ignition and apparent flame structures. The pure coal 400 particle (215–255 µm) ignited after 30 ms, whereas the wood particle ignited after 48 ms. The 401 single particles with fuel mixtures ignite in almost the same way as does the coal particle, 402 which is attributed to the coal ignition characteristics. To see this in more detail, the results 403 404 for ignition and combustion times are discussed in Fig. 12-15, where it can be seen that increasing the coal blending ratio has an impact on large elongated flames with their high 405 intensity. 406



407

Fig. 9. Sequential combustion-process images of four solid particles ($d_{255-300 \ \mu m}$, $T_g = 1340 \ K$ and 10% O₂ concentration). After leaving the tip of the injector, each single particle is marked at intervals of 6 ms as it forms an apparent volatile flame from initial ignition.

412 **3.3 Ignition delay, volatile combustion time and flame characteristics**

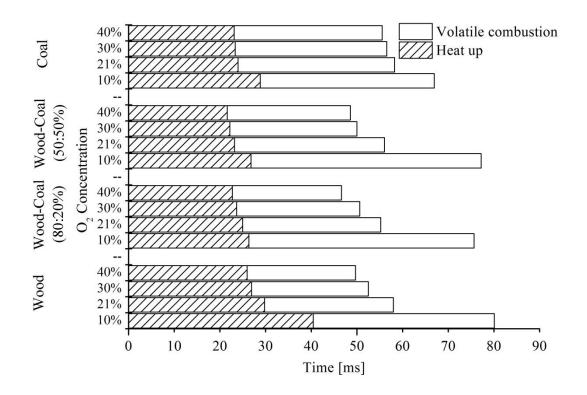
Figure 10 shows the average heat-up (or ignition) and volatile combustion durations for the 413 four particles from 20 of each particle under different oxygen concentrations at 1,340 K. The 414 standard deviation of each test is less than 10% of the mean values from the quantitative 415 analysis. The heat-up time is measured from when the particle is injected until the apparent 416 417 volatile ignition. This ignition might be different from homologous ignition, because a light gas phase emits a very weak light. Therefore, this experiment presents only the apparent 418 419 volatile ignition with soot luminosity. The results show that, under 10% oxygen, the coal particle and the two co-firing particles have shorter ignition delays (26-30 ms) than that of 420

421 the wood particle (40 ms).

434

Under all oxygen levels, the differences in apparent volatile ignition between particles (except for the pure wood particles) are not significant because of early ignition, based on apparent volatile ignition of the coal particle. The volatile combustion time of the co-firing particles is almost the same as that of wood. In other words, the different ignition and volatile combustion times of the co-firing particles can be attributed to both coal and wood, respectively.

The apparent flame characteristics of the four particles at $215-255 \mu m$ are explained under 10, 21 and 30% O₂ as shown in Fig. 11. Overall, the size of the volatile flames decreases with the shorter combustion time as the oxygen concentration increases. This result shows that the four particles have distinguishable flame structures in terms of size and intensity under all oxygen levels. Their flame size and intensity are explained by a quantitative analysis in section 3.4.



435 Fig. 10. Measurement of average duration of heat-up and volatile combustion, obtained from

four particles (255–300 μ m) under different oxygen concentrations at 1,340 K. This figure shows different burning characteristics between the wood and coal particles and the two mixed particles.

439

1 10 % O₂ Heat up regime 33 m 42m51 m 1 1 1 1 . . 33 m 42 m 51 m P 2 0 440 . . 0 . 21 % O₂ Heat up regime 51 m 33 m 42 m 60 m C . 60 m 39 n 48m 57 m 57 m 39 ms 48 ms 66 m 441 0 0 30% O₂ Heat up regin 24 ms 51 n • • ø 5 24 m 33 m 51 m 42 m 33 m 42 ms 24 m 442

Fig. 11. Burning particles at 215–255 μm under 10%–30% O₂ concentrations at 1,340 K: (a)
coal, (b) 20:80 coal/wood, (c) 50:50 coal/wood and (d) wood. The particles are captured over
time intervals of 4 ms. The coal particle has the largest and brightest flame of the four.

446

447 **3.4 Effect of particle types over time**

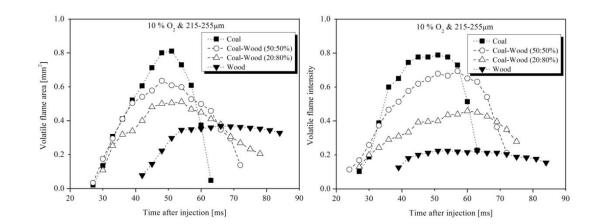
The results shown in Fig. 12 suggest that increasing the coal blending ratio under 10 and 21% oxygen concentrations has a significant effect on the enveloped flame of each particle in the time domain. The size and intensity of the flames around particles (215-255 μ m) were measured from volatile ignition to flame extinction. However, we cannot detect the whole volatile combustion in particles (300-350 μ m) as explained before. Particle combustion events are associated with a high peak when the maximum volume of volatiles and soot particles are in the flame. This can also be seen in the nearly symmetrical profile during combustion [17]. From this result, the profiles of the two particles with fuel mixtures are seen to lie between the coal and wood profiles. The peak flame size does not correspond entirely with the peak flame intensity, although they have similar tendencies over the same period. The peak of volatile flame area occurs marginally before the intensity peak.

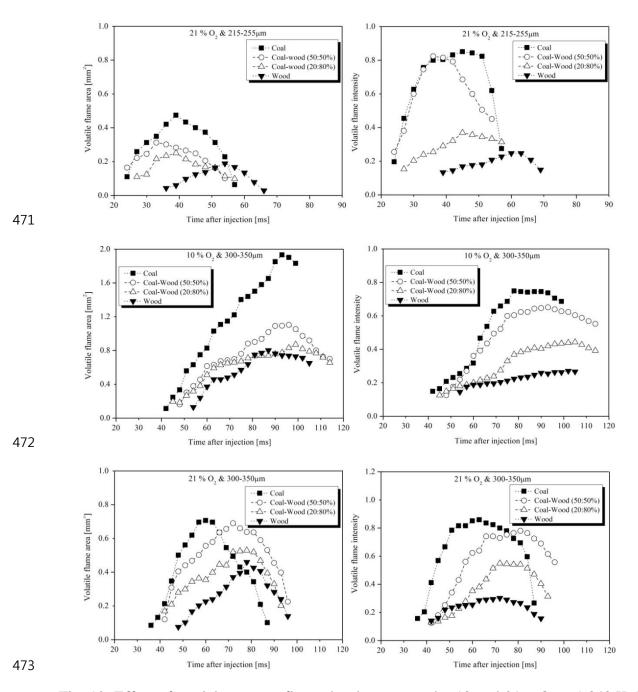
459 Figure 13 shows the variation in maximum flame area and intensity under 21% O₂ at 1,340 K with the wood-coal blending ratio. The imagined line between wood and coal 460 particles is drawn in order to compare their measured magnitudes. To obtain the results, the 461 462 flame parameters were averaged over 20 particles for each test. The flame area of the 50:50 coal/wood particle differed from the prediction of the imagined line by 0.03 mm². Both the 463 20:80 and 50:50 coal/wood particles have a higher intensity compared with the trend in flame 464 area. Also, there was little difference in intensity between the 50:50 coal/wood and pure coal 465 particles under 21% O₂. 466

467

468

469





474 Fig. 12. Effect of particle type on flame development under 10 and 21% O_2 at 1,340 K. The 475 figures show a peak in the flame parameters as a function of time, and distinguishable flame 476 size and intensity between particles are measured at 215–255 μ m and 300–350 μ m.

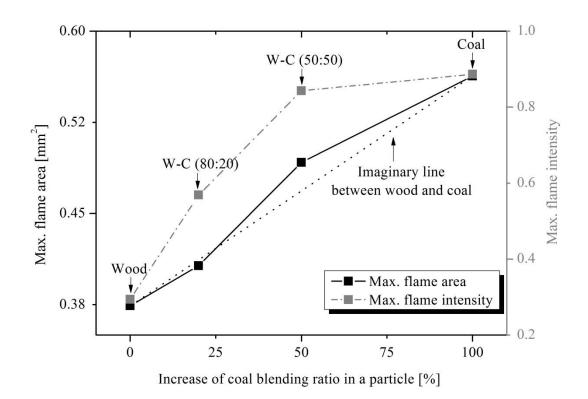


Fig. 13. Overall profiles of average flame parameters with particle coal-blending ratio (255–300 μm) under 21% O₂ at 1,340 K. This figure shows how flame intensity and area increase
with reference to an imagined line between wood and coal.

481

482 **3.5 Effect of oxygen concentration with particle type as a function of time**

The four different types of particle in the particle size group of 215–255 µm were burned 483 under 10%–40% O₂. Fig. 14 shows the temporal flame profiles for the effect of oxygen 484 concentration. The flame area decreased with shorter volatile combustion time as the oxygen 485 concentration was increased. A dramatic decrease of this flame parameter for particles at 486 487 255-300 µm was observed between 10 and 21 % oxygen, as shown in Fig. 15(a). Particles burning under low oxygen concentration have an elongated flame. The flame areas of coal, 488 489 50:50 coal/wood, 20:80 coal/wood and wood under 21%-30% O₂ decreased by 490 approximately 44, 40, 59 and 79%, respectively. As for the combustion mode, particles 491 containing 80% or more of wood have imperceptible volatile flames with low luminosity at
492 high oxygen concentrations, and simultaneous homogeneous and heterogeneous combustion
493 occurs on the surface. Hence, coal blending plays an important role in stable flame structures
494 and sequential combustion processes even under high oxygen concentrations.

495

Overall, the flame intensity of the four particles at 255–300 µm present the effect of oxygen 496 concentration as shown in Fig. 15(b). The flame intensities on all the particles were 497 comparatively high under 21% oxygen. However, the peak flame intensity of the coal 498 499 particles was reached at 30% oxygen concentration, whereas the pure wood particle and particles with wood mixtures have their highest flame intensities at 21% O₂. This 500 phenomenon is likely attributable to the volume fraction of soot in the flame. The normalised 501 502 flame intensities of particles with high wood mixtures decreased dramatically between 21 % and 30 % O₂. Khatami et al. [15] reported that the highest peak soot volume fraction of 503 bituminous coal occurred in 40 % O. The different profiles between two results are related to 504 the particle size, environmental conditions and reaction rate. 505

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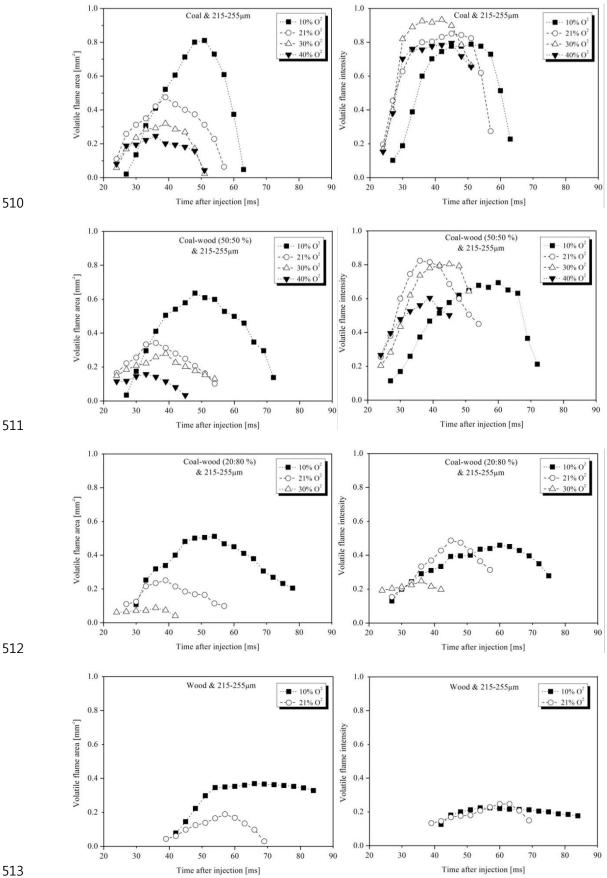


Fig. 14. Effect of oxygen concentration (10%-40%) on the flame characteristics of the four 514

515 particles (215–255 μ m) at 1,340 K. The graphs illustrate the variation of volatile flame area 516 and intensity with surrounding O₂.

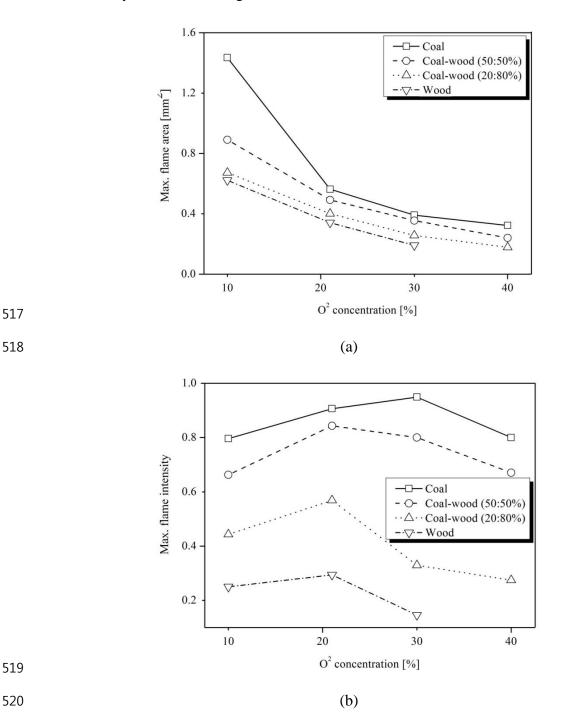


Fig. 15. Overall profiles of average flame area and intensity of the four particles $(255-300 \ \mu m)$ with O₂ concentration. These graphs show a wide variation in flame area for the four particles under 10% O₂; wood and blended particles (not coal particles) have a peak flame intensity at 21% O₂.

525 **3.6 Effect of particle size**

Figure 16 shows superimposed visible flame appearances for particle sizes in the ranges of 526 215–255 µm and 300–350 µm. All the experiments were performed under 10% oxygen 527 concentration at 1,340 K. After injection, the burning particles were captured at 4-ms time 528 intervals. The increase of approximately 100 µm in the size of the four particle types clearly 529 affected the ignition delay. The trajectories of the 300–350-µm particles dropped slightly after 530 injection and then went upwards until burnout because of the relatively high particle mass. In 531 532 the experiment, the particle size group of 355–425 µm dropped slightly after injection and then lifted toward the top until burnout. Larger particle sizes fall to the bottom to be 533 incompletely burned without volatile combustion, which was explained by Mock et al. [17]. 534 535 Figure 17 shows the average flame parameters for 20 particles of 215–255 µm and 300–

355 Figure 17 shows the average finite parameters for 25 particles of 215–255 µm and 500– 356 350 µm under 21% oxygen concentration. This shows that the increase in flame size for all particles was very steady between the two particle-size groups. However, the increased flame intensity is a little different between coal particles and particles with a high coal-blending ratio. The coal and 50:50 coal/wood particles also show very similar flame intensity variations with particle size, which may be attributed to a sufficiently high soot particle volume fraction in the particles.

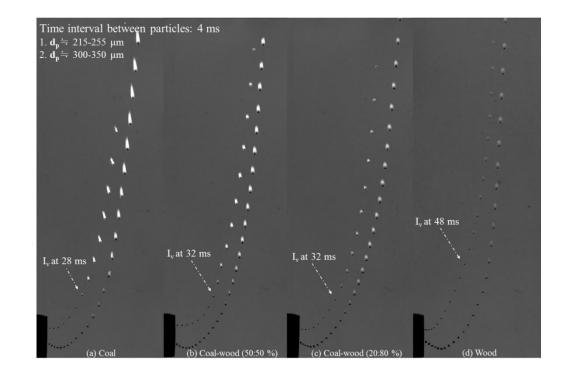


Fig. 16. Superposition of four different particles of 215–255 µm and 300–350 µm at time
intervals of 4 ms entrained in hot gas streams of 1,340 K under 10% oxygen concentration.
The tendencies of particles of different sizes are similar between the two particle size groups.
However, the flames on the 300–350 µm particles grow because of the longer ignition delay.

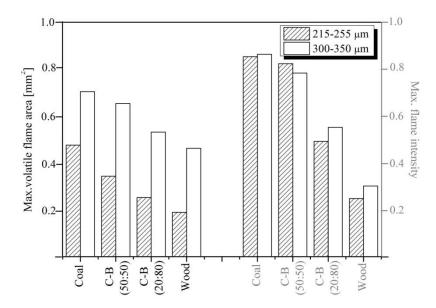
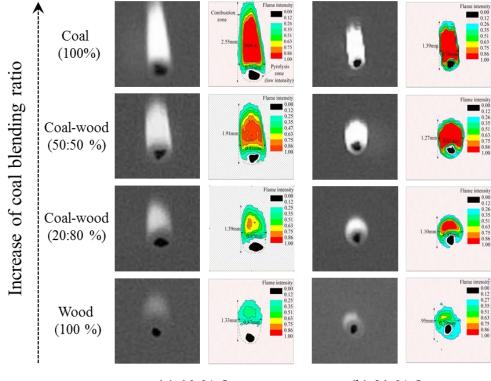


Fig. 17. Average flame area and intensity of four different particles with particle size (215– 255 μ m and 300–350 μ m) under 21% O₂ concentration. The graphs show a regular decreasing trend in volatile flame area.

552 **3.7 Flame aspect ratio of co-firing particles**

The shapes of the flames on the four particle surfaces under 10 and 21% oxygen 553 554 concentrations can be represented in terms of the flame aspect ratio. Figures 18(a) and (b) illustrate the measurement of flame height, length and intensity after image post-processing, 555 which may be an effective way to define the characteristics of flame shape. One of the results 556 557 is that a relatively large pyrolysis zone for fuel vaporisation and reactions was observed 558 clearly under low oxygen concentration, showing a transparent coloured boundary layer. The 559 flame aspect ratio increases rapidly for burning particles with a 50% coal blending ratio. However, wood and 20:80 coal/wood particle have aspect ratios of approximately 1.2, which 560 suggests a more spherical flame. An elongated flame can be attributed to soot phase in a 561 562 particle. Coal particles, which contain more fixed carbon and less volatile matter, have the 563 highest visible flame aspect ratio of 3.4 under the lowest oxygen concentration.

564



(a) 10 % O₂

(b) 21 % O₂

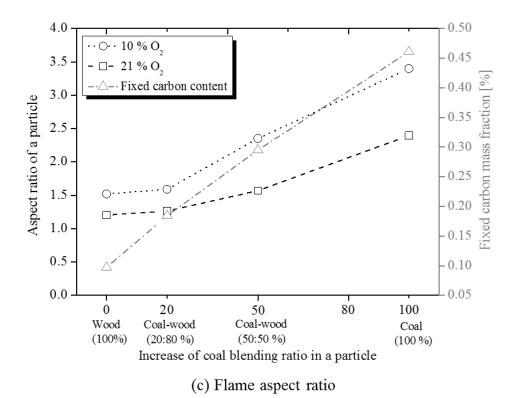


Fig. 18. Flame structure in terms of aspect ratio obtained from volatile flames of four
particles under 10 and 21% O₂ concentrations by image post-processing.

569 **4. CONCLUSIONS**

In the study, pulverised single solid-fuel particles with different fuel mixtures were burned 570 in a lab-scale entrained-flow reactor. These were compared with the burning of pure coal and 571 pure wood particles under identical environmental conditions. The four particles differed in 572 573 relation to their chemical and physical characteristics, which in turn affected the flame characteristics. The main objectives were to assess the burning behaviours of mixed single 574 particles for co-firing, focusing on ignition, flame area and intensity, and combustion time, 575 576 from direct observation. Quantitative analysis of these combustion parameters was presented along with a description of the sequential combustion process of the particles under various 577 oxygen concentrations and for different particle sizes. The burning of particles under both 578 slow and rapid heating was also discussed in relation to the similarities and differences of 579

their combustion behaviours. The significant conclusions of this study are as follows:

581

Single particles were observed in sequential combustion processes such as heat-up,
 ignition and volatile and char combustion. Simultaneous volatile and char combustion
 has been observed in a particle with high wood mixture under enhanced oxygen
 concentrations.

- 586
 2. The conical flame shape on co-firing particles was affected by an increase in the coal
 587 mixture ratio. This was because the high fixed carbon content and H/C ratio of coal
 588 generate a large soot flame with a high aspect ratio and flame intensity.
- 3. The pulverised particles are burned out in a few hundred milliseconds under rapid
 heating; coal particles ignited sooner than the apparent ignition of wood particles. As
 a result, the ignition of pulverised co-firing particles is attributed to the coal ignition
 characteristics. However, the volatile combustion time plays an important role in the
 high volatile matter content of the wood mixtures.
- 4. Of relevance to co-firing combustion is the fact that particles with coal/wood
 mixtures improve the radiative heat energy from the flame parameters compared with
 wood flames: +9% and +32% for the size and 93% and 187% for the intensity of
 20:80 coal/wood and 50:50 coal/wood, respectively, under 21% O₂. This may affect
 the stable diffusion flame, the fast homogeneous ignition and the flame stability to the
 boiler efficiency.
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- 604

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