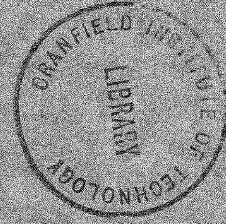
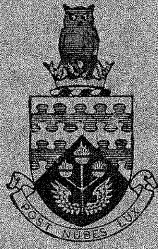


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CoA Note No. 67



THE COLLEGE OF AERONAUTICS  
CRANFIELD

VORTEX TUBE PERFORMANCE DATA SHEETS

by

R. Westley

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THE COLLEGE OF AERONAUTICS

C R A N F I E L D



Vortex Tube Performance Data Sheets

-by-

R. Westley, D.C.Ac.

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1. Introduction

Experiments to determine the effect of various operating and design parameters on the performance of a vortex tube are described in College of Aeronautics Note No. 30. The present note gives additional performance data on this tube and should be used as a supplement to Note No.30.

These data sheets are intended as an aid to the estimation of the performance of vortex tubes and to the design of vortex tubes with given characteristics. In particular, the present note provides the optimum values of the vortex tube parameters which give maximum temperature drops.

Figures 12-53 give temperature drop and cold mass flow ratio characteristics, figures 54-61 give the cold mass flow ratios at maximum temperature drop whilst figures 62-65 give the optimum inlet and cold outlet sizes for maximum temperature drop.

2. NOTATION

$p_i$	inlet pressure
$p_c$	cold outlet pressure
$p_h$	hot outlet pressure
$\Delta p_i$	pressure differential across inlet flow meter
$\Delta p_c$	" " " cold " "
$\Delta p_h$	" " " hot " "
$T_i$	inlet temperature
$T_c$	cold "
$T_h$	hot "
$\Delta T_c$	$\equiv (T_i - T_c)$

$$\left(\frac{\Delta T_c}{T_i}\right)_1 \equiv \frac{\Delta T_c}{T_i} \text{ for } \frac{\partial \frac{\Delta T_c}{T_i}}{\partial \mu} = 0.$$

$$\left(\frac{\Delta T_c}{T_i}\right)_2 \equiv \frac{\Delta T_c}{T_i} \text{ for } \frac{\partial \frac{\Delta T_c}{T_i}}{\partial \mu} = 0, \quad \frac{\partial \left(\frac{\Delta T_c}{T_i}\right)_1}{\partial d_c} = 0.$$

$$\left(\frac{\Delta T_c}{T_i}\right)_3 \equiv \frac{\Delta T_c}{T_i} \text{ for } \frac{\partial \frac{\Delta T_c}{T_i}}{\partial \mu} = 0, \quad \frac{\partial \left(\frac{\Delta T_c}{T_i}\right)_1}{\partial d_c} = 0, \quad \frac{\partial \left(\frac{\Delta T_c}{T_i}\right)_2}{\partial d_i} = 0.$$

$$\frac{T_c}{T_i} \equiv 1 - \left(\frac{\Delta T_c}{T_i}\right)_3$$

$$1 - \frac{\Delta T_c \text{ isentropic}}{T_i} = \left(\frac{p_c}{p_i}\right)^{\frac{\gamma-1}{\gamma}}, \quad \gamma = 1.40.$$

$$\Delta T_{c2} \equiv \left(\frac{\Delta T_c}{T_i}\right)_2 \times T_i.$$

$$\Delta T_{c3} = \left( \frac{\Delta T_c}{T_i} \right)_3 \times T_i.$$

D diameter of vortex tube

$d_c$  diameter of cold outlet

$d_i$  equivalent inlet nozzle diameter =  $\sqrt{\frac{4 \text{ Total inlet area}}{\pi}}$

$l_i$  length of inlet slot

$$\left( \frac{d_c}{D} \right)_1 = \frac{d_c}{D} \text{ for } \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)}{\partial \mu} = 0, \quad \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)_1}{\partial \frac{d_c}{D}} = 0.$$

$$\left( \frac{d_c}{D} \right)_2 = \frac{d_c}{D} \text{ for } \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)}{\partial \mu} = 0, \quad \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)_1}{\partial \frac{d_c}{D}} = 0, \quad \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)_2}{\partial \frac{d_i}{D}} = 0.$$

$$\left( \frac{d_i}{D} \right)_1 = \frac{d_i}{D} \text{ for } \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)}{\partial \mu} = 0, \quad \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)_1}{\partial \frac{d_c}{D}} = 0, \quad \frac{\partial \left( \frac{\Delta T_c}{T_i} \right)_2}{\partial \frac{d_i}{D}} = 0.$$

R hot valve setting (revolutions)

$$\left( \frac{A_c}{A} \right)_2 = \left\{ \left( \frac{d_c}{D} \right)_2 \right\}^2$$

$$\left( \frac{A_i}{A} \right)_1 = \left\{ \left( \frac{d_i}{D} \right)_1 \right\}^2.$$

$\mu$  cold mass flow ratio =  $\frac{\text{cold mass flow}}{\text{inlet mass flow}}$

$\mu_1$  cold mass flow ratio at  $\frac{\Delta T_c}{T_i} = \left( \frac{\Delta T_c}{T_i} \right)_1$

$\mu_2$  " " " " " " =  $\left( \frac{\Delta T_c}{T_i} \right)_2$

$\mu_3$  " " " " " " =  $\left( \frac{\Delta T_c}{T_i} \right)_3$

3. LIST OF FIGURES & DATA SHEETS

- Fig.1. Inlet chamber and cold outlet.
- Fig.2. Hot air outlet valve.
- Fig.3. Exploded view of cold outlet diaphragm, inlet nozzle component and tube.
- Fig.4. Alternative vortex tube components.
- Fig.5. Cold outlet diaphragm.
- Fig.6. Inlet nozzles
- Fig.7. General view of vortex tube apparatus.
- Fig.8. General layout of vortex tube apparatus.
- Fig.9. Vortex tube apparatus: General assembly, front view,
- Fig.10. Vortex tube apparatus: General assembly, plan view.
- Fig.11. Thermocouple probe
- Fig. 12-41. Variation of temperature drop ratio  $\left(\frac{\Delta T_c}{T_i}\right)$  with cold mass flow ratio ( $\mu$ ) for various pressure ratios  $\left(\frac{P_i}{P_c}\right)$

$\frac{d_c}{D} = \frac{d_i}{D} =$	.266	.376	.461	.532	.595
.167	-	Fig.18.	Fig.24.	Fig.31.	Fig.37.
.250	Fig.12.	Fig.19.	Fig.25.	Fig.32.	Fig.38
.333	Fig.13.	Fig.20.	Fig.26.	Fig.33.	Fig.39.
.417	Fig.14.	Fig.21.	Fig.27.	Fig.34.	Fig.40.
.500	Fig.15.	Fig.22.	Fig.28.	Fig.35.	Fig.41.
.583	Fig.16.	-	Fig.29.	Fig.36.	-
.667	Fig.17.	Fig.23.	Fig.30.	-	-

Figs. 42-46. Variation of maximum temperature drop ratio  $\left(\frac{\Delta T_c}{T_i}\right)_1$  with cold outlet diameter ratio  $\left(\frac{d_c}{D}\right)$  for various pressure ratios  $\left(\frac{p_i}{p_c}\right)$ .

$\frac{d_i}{D} =$	.266	.376	.461	.532	.595
—	Fig.42.	Fig.43.	Fig.44.	Fig.45.	Fig.46.

Fig. 47. Variation of temperature drop ratio  $\left(\frac{\Delta T_c}{T_i}\right)_2$  with pressure ratio  $\left(\frac{p_i}{p_c}\right)$  for optimum cold outlet diameter, optimum value setting and given inlet size  $\left(\frac{d_i}{D}\right)$ .

Fig. 48. Variation of temperature drop ratio with inlet size for optimum cold outlet and valve setting,  $\left(\frac{p_i}{p_c} = 1.5, 2.0, 3.0, 4.0, 5.0, 6.0, 7.0\right)$ .

$$\left(\frac{\Delta T_c}{T_i}\right)_2 \sim \frac{d_i}{D}$$

Fig. 49. Variation of temperature drop efficiency with pressure ratio at optimum cold outlet size, optimum valve setting and fixed inlet diameter.

$$\left(\frac{d_i}{D} = .266, .376, .461, .532, .595\right)$$

$$\frac{\Delta T_{c2}}{\Delta T_{c \text{ ISEN}}} \sim \frac{p_i}{p_c}$$

Fig. 50. Variation of refrigeration efficiency with pressure ratio  $\left(\frac{p_i}{p_c}\right)$  at optimum valve setting and cold outlet diameter for maximum temperature drop ratio

$$\left[ \frac{\mu_2 \left(\frac{\Delta T_c}{T_i}\right)_2}{\left(\frac{\Delta T_c}{T_i}\right)_{\text{ISEN}}} \right]$$

Fig. 51. Variation of temperature drop ratio  $\left(\frac{\Delta T_c}{T_i}\right)_3$  and refrigeration ratio  $\left[\mu_3\left(\frac{\Delta T_c}{T_i}\right)_3\right]$  with pressure ratio at maximum temperature drop and optimum conditions.

Fig 52. Power law variation of temperature drop ratio and refrigeration ratio with pressure ratio  $\left(\frac{P_i}{P_c}\right)$  at maximum temperature drop.

Fig. 53. Variation of temperature drop efficiency  $\left[\frac{\left(\frac{\Delta T_c}{T_i}\right)_3}{\left(\frac{\Delta T_c}{T_i}\right)_{ISEN}}\right]$  and refrigeration efficiency  $\frac{\mu_3\left(\frac{\Delta T_c}{T_i}\right)_3}{\left(\frac{\Delta T_c}{T_i}\right)_{ISEN}}$  with pressure ratio  $\left(\frac{P_i}{P_c}\right)$  at maximum temperature drop.

Fig. 54-58. Variation of cold mass flow ratio  $\left(\mu_1\right)$  with cold outlet diameter ratio  $\left(\frac{d_c}{D}\right)$  at optimum valve setting

$\frac{d_i}{D} =$	.266	.376	.461	.532	.595
	Fig.54.	Fig.55.	Fig.56.	Fig.57.	Fig.58.

Fig. 59. Variation of cold mass flow ratio  $\left(\mu_2\right)$  with inlet diameter ratio  $\left(\frac{d_i}{D}\right)$  at optimum valve setting and optimum cold outlet diameter.

Fig. 60. Variation of cold mass flow ratio  $\left(\mu_2\right)$  with pressure ratio  $\left(\frac{P_i}{P_c}\right)$  at optimum valve setting and optimum cold outlet diameter.

Fig. 61. Variation of cold <sup>mass</sup> flow ratio with pressure ratio for optimum valve setting, optimum cold outlet diameter and optimum inlet diameter

$$\mu_3 \sim \frac{P_i}{P_c}$$

Fig. 62. Variation of optimum cold outlet diameter with inlet diameter  $\left(\frac{P_i}{P_c} = 1.5, 2, 3, 4, 5, 6, 7\right)$ .

$$\left(\frac{d_c}{D}\right)_1 \sim \frac{d_i}{D}$$

Fig. 63. Variation of optimum cold outlet diameter with pressure ratio for fixed inlet diameters,  $\left(\frac{d_i}{D} = .266, .376, .461, .532 \text{ \& } .595\right)$ .

$$\left(\frac{d_c}{D}\right)_1 \sim \frac{P_i}{P_c}$$

Fig. 64. Optimum inlet and cold outlet diameters for maximum temperature drop.

$$\left(\frac{d_i}{D}\right)_1 \sim \frac{P_i}{P_c}$$

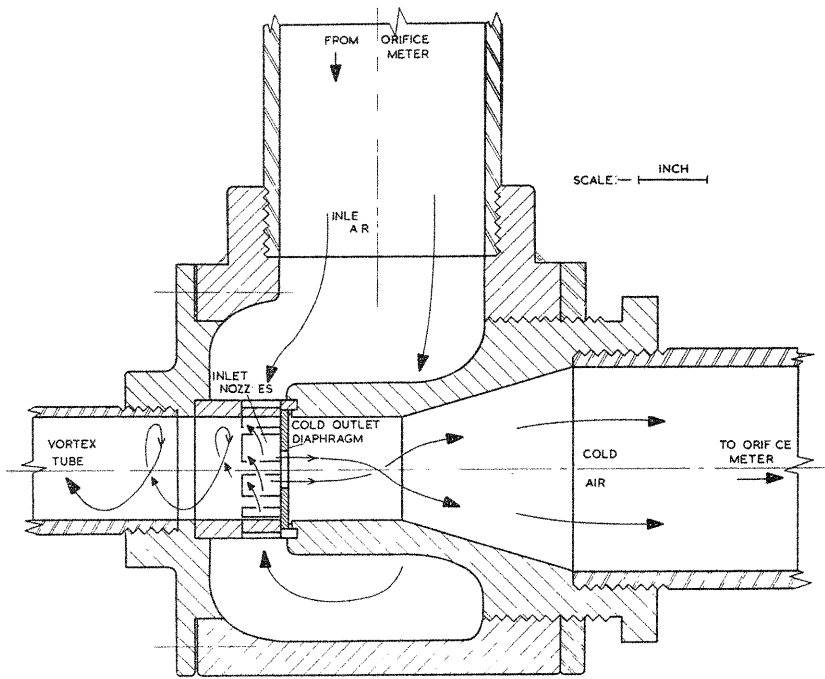
$$\left(\frac{d_c}{D}\right)_2 \sim \frac{P_i}{P_c}$$

Fig. 65. Optimum inlet and cold outlet areas for maximum temperature drop.

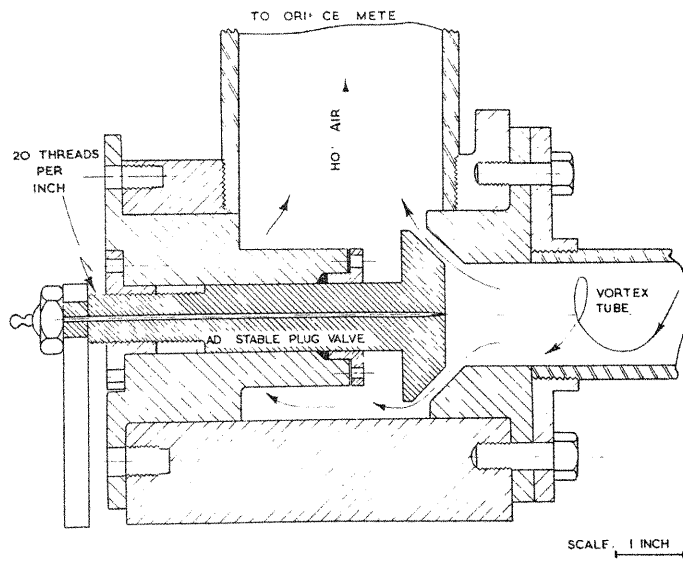
$$\left(\frac{A_i}{A}\right)_1 \sim \frac{P_i}{P_c}$$

$$\left(\frac{A_c}{A}\right)_2 \sim \frac{P_i}{P_c}$$





INLET CHAMBER AND COLD OUTLET  
FIG. 1



HOT AIR OUTLET VALVE  
FIG. 2.

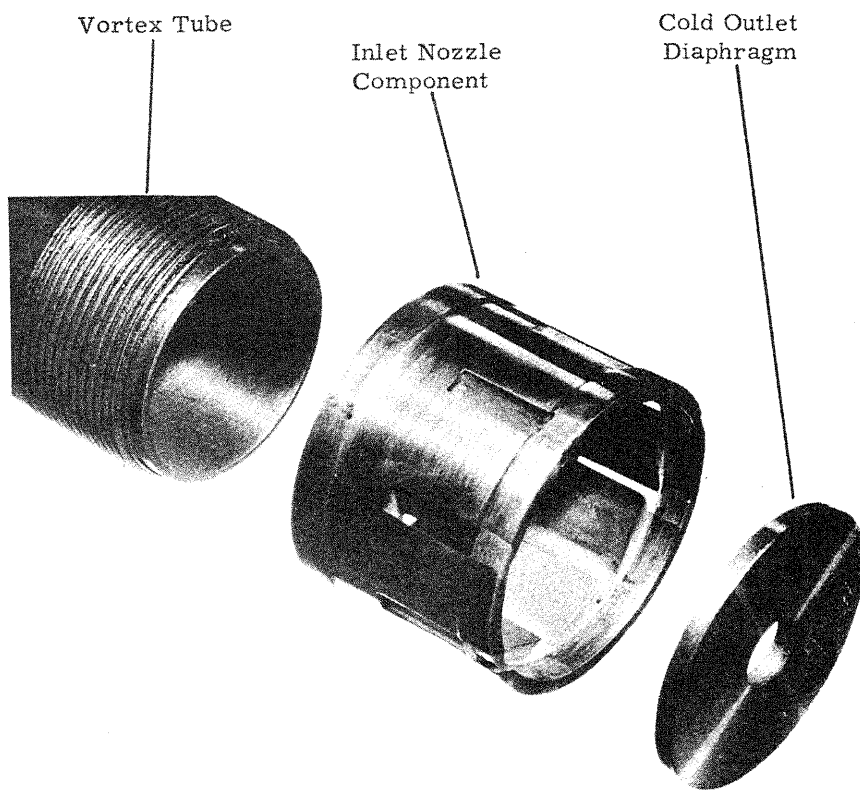


FIGURE 3. EXPLODED VIEW OF COLD OUTLET DIAPHRAGM,  
INLET NOZZLE COMPONENT AND TUBE.

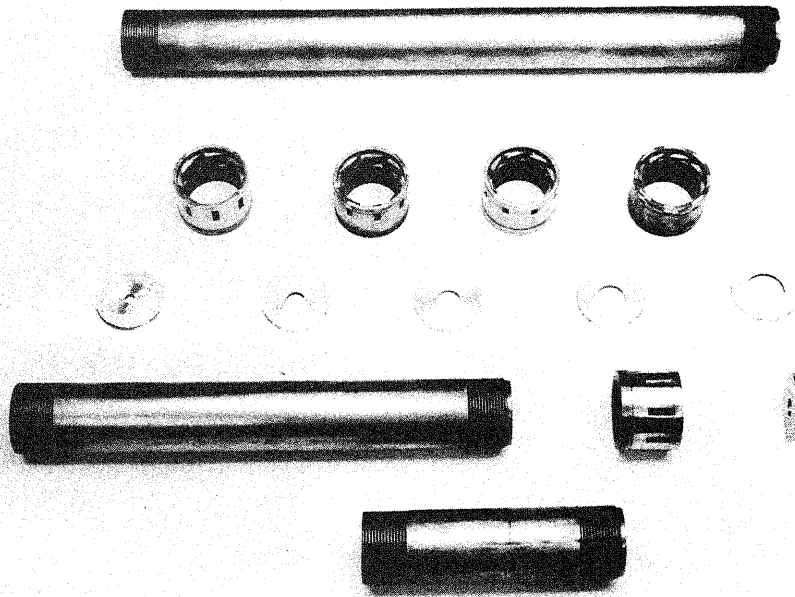


FIGURE 4 ALTERNATIVE VORTEX TUBE COMPONENTS

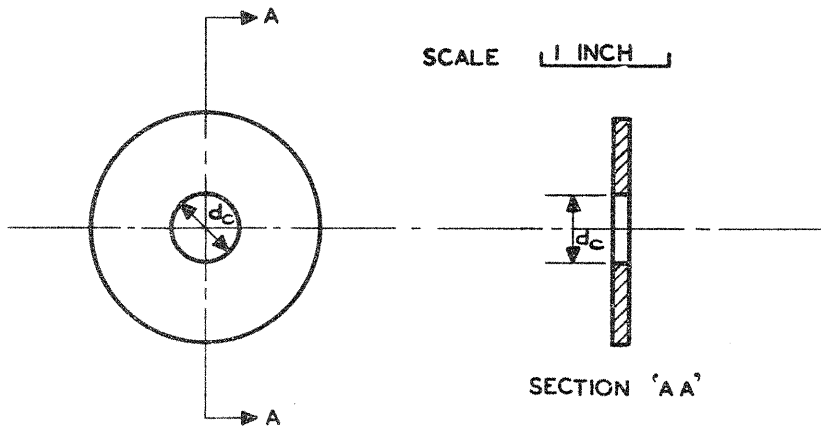


FIGURE 5. COLD OUTLET DIAPHRAGM

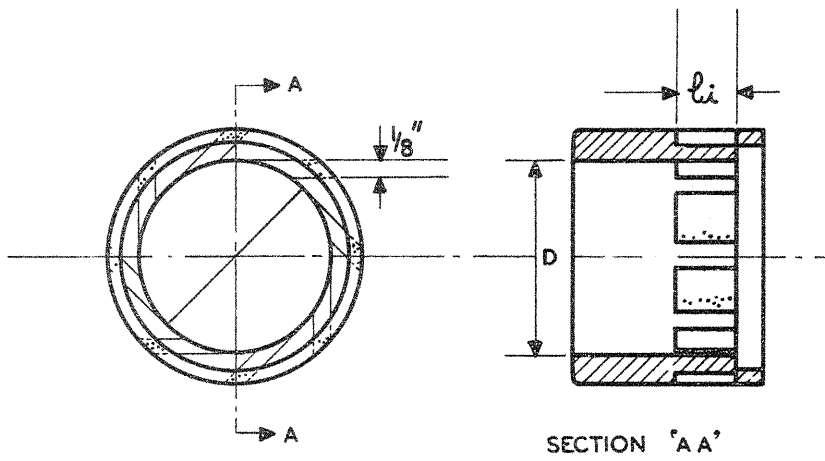


FIGURE 6. INLET NOZZLES

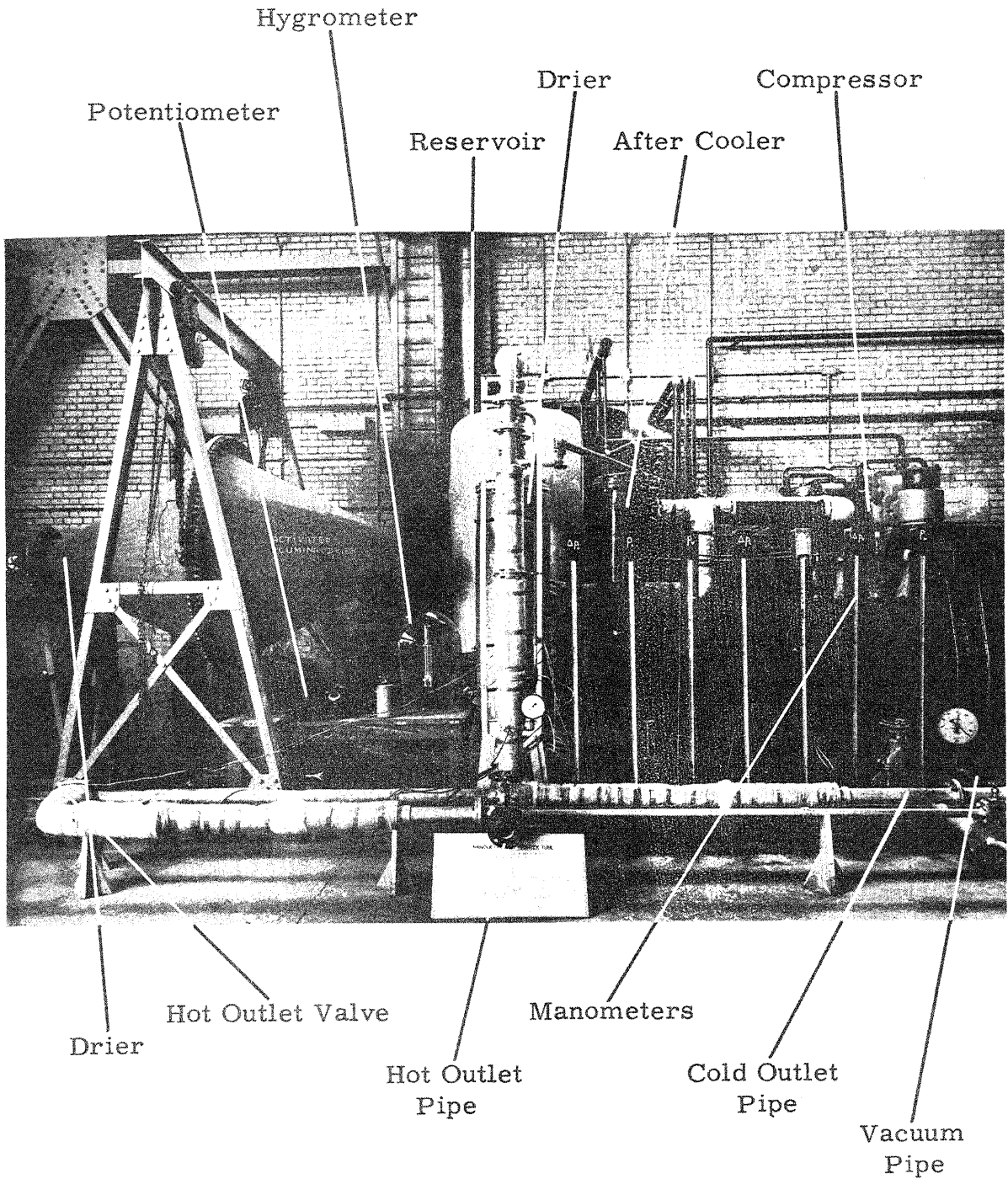


FIGURE 7. GENERAL VIEW OF VORTEX TUBE APPARATUS

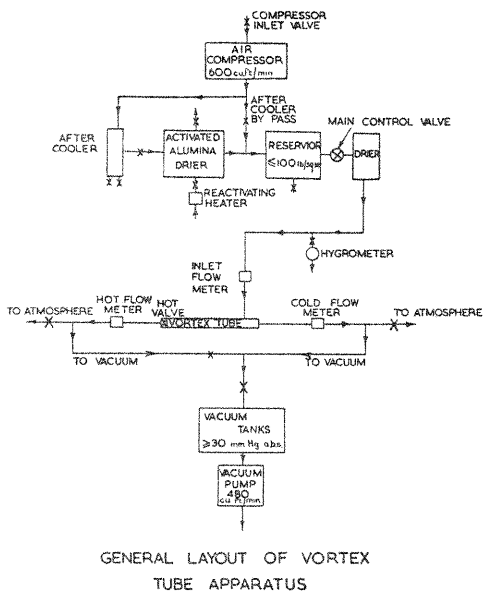
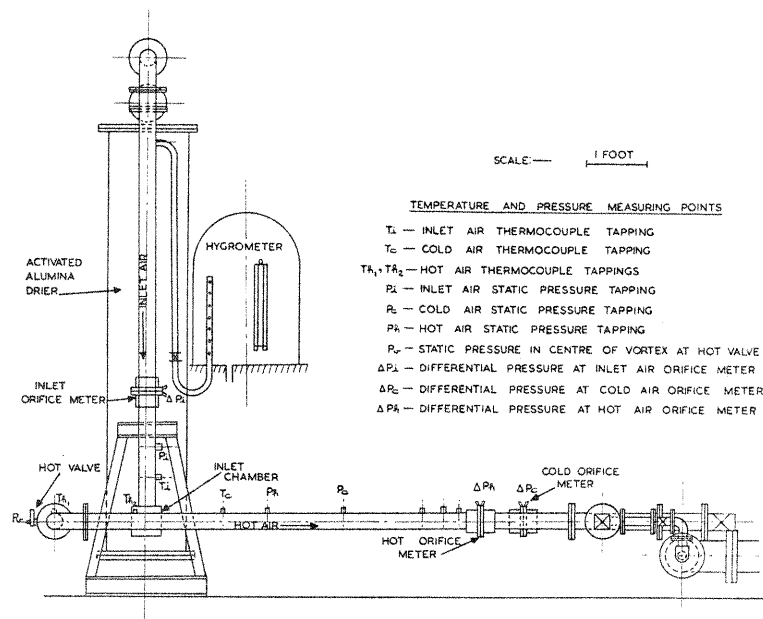
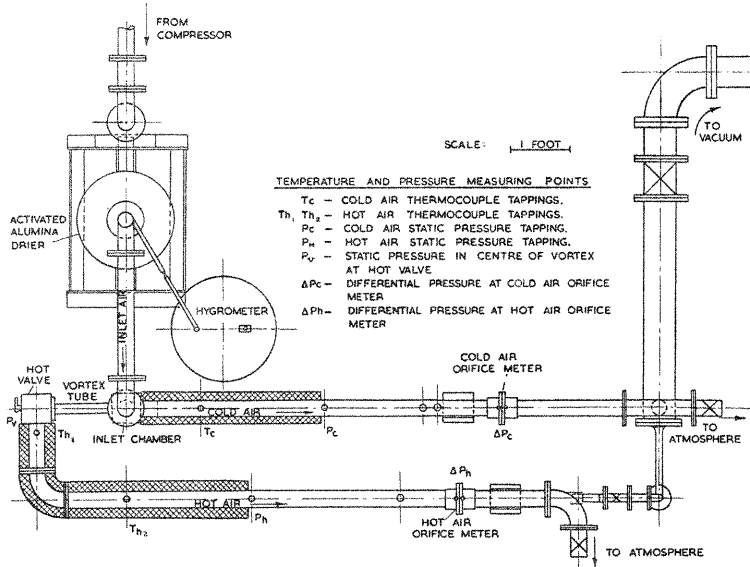


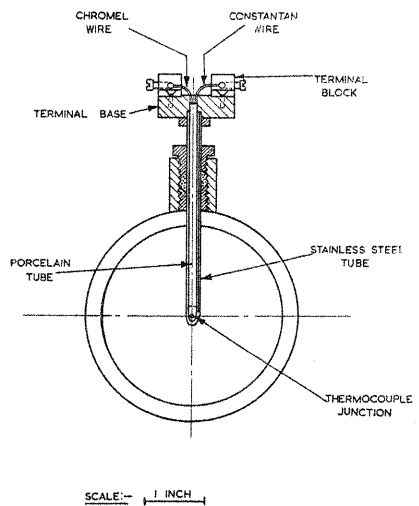
FIG. 8.



VORTEX TUBE APPARATUS GENERAL ASSEMBLY  
FRONT VIEW.  
FIG. 9.



VORTEX TUBE APPARATUS GENERAL ASSEMBLY  
PLAN VIEW.  
FIG. 10.



THERMOCOUPLE PROBE.  
FIG. 11.

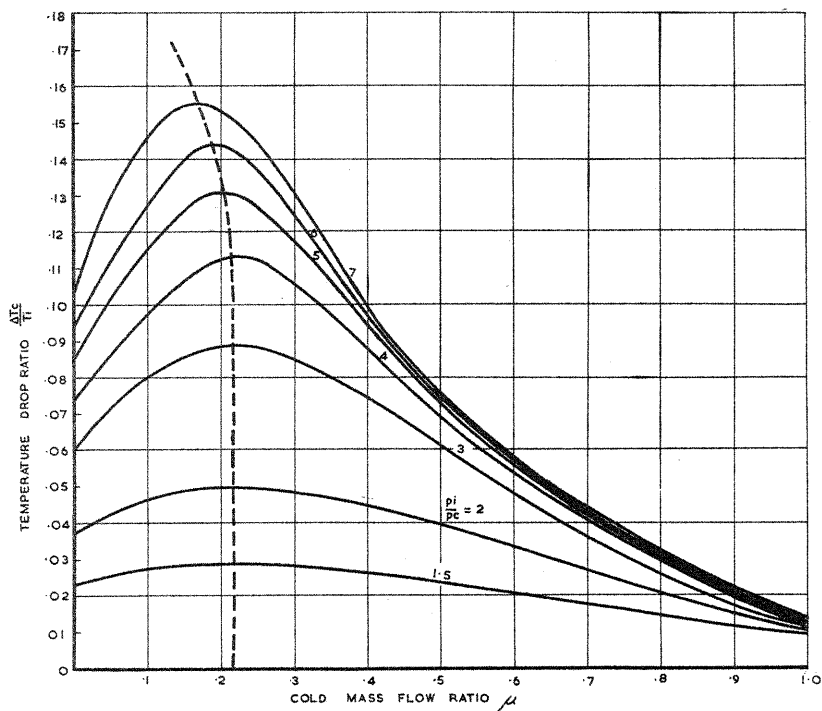


FIG. 12 VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .250, \frac{d_1}{D} = .266$

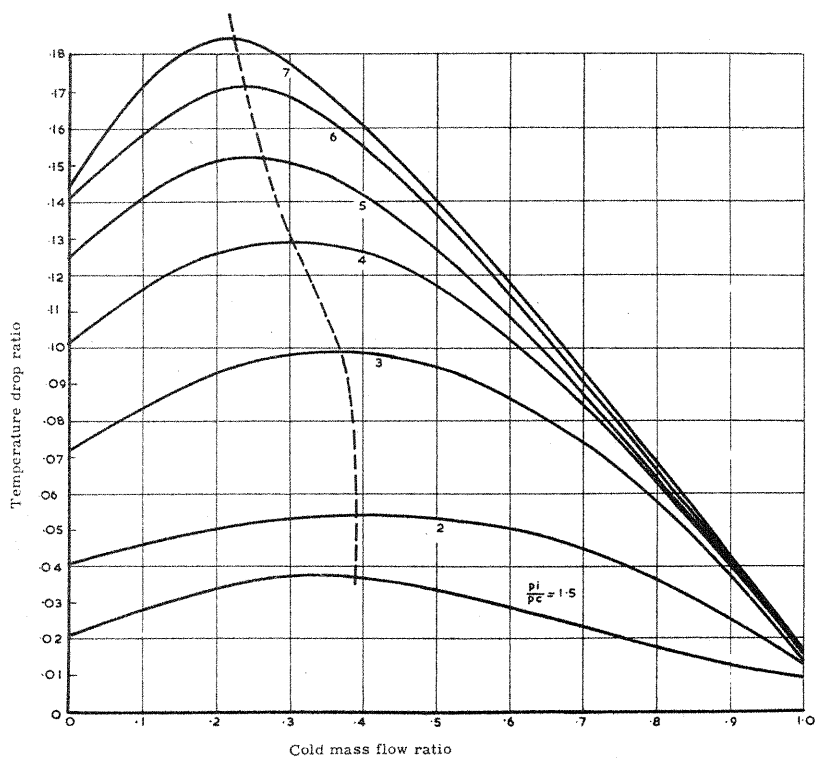


FIG. 13. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .333, \frac{d_1}{D} = .266$

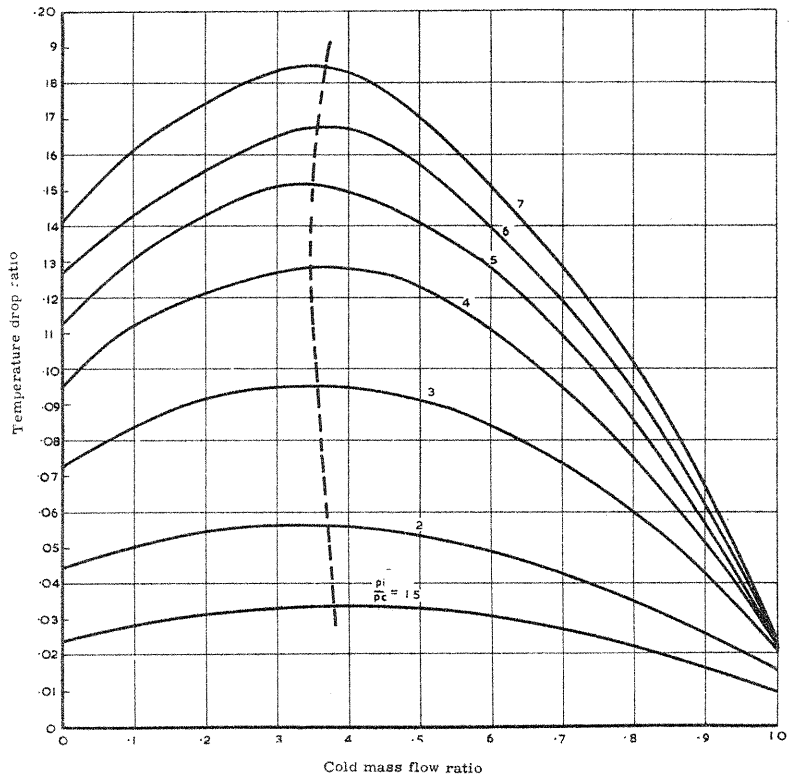


FIG. 14 VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = 41$ ,  $\frac{d_1}{D} = 2.6$

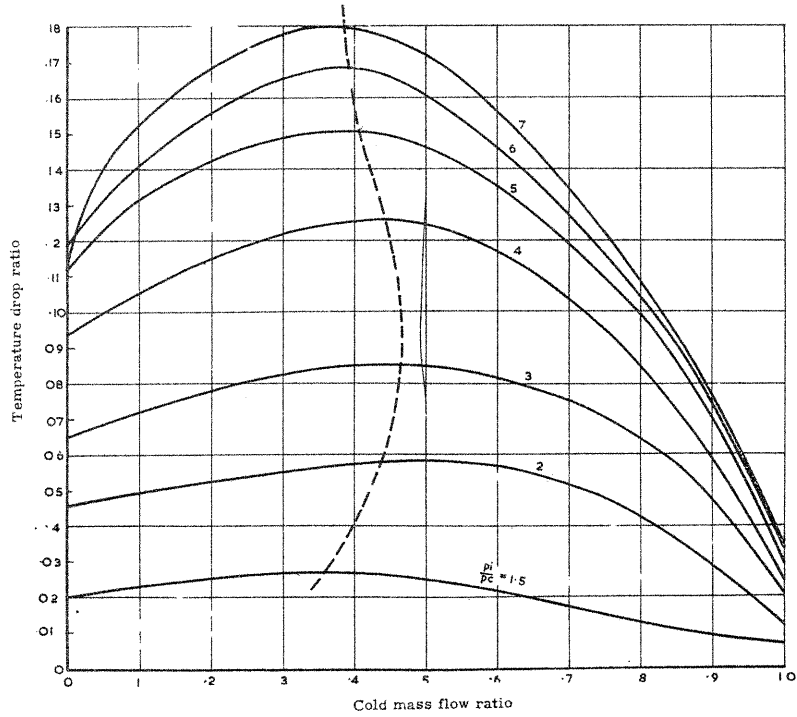


FIG. 15 VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = 500$ ,  $\frac{d_1}{D} = 266$

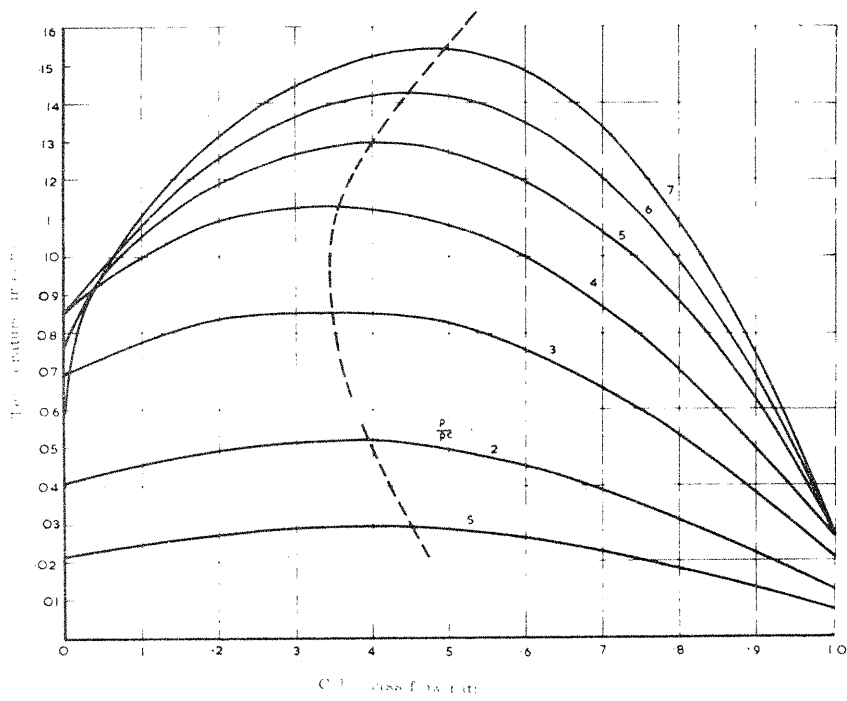


FIG. 16. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = 0.64$ ,  $\frac{d_h}{D} = 0.266$

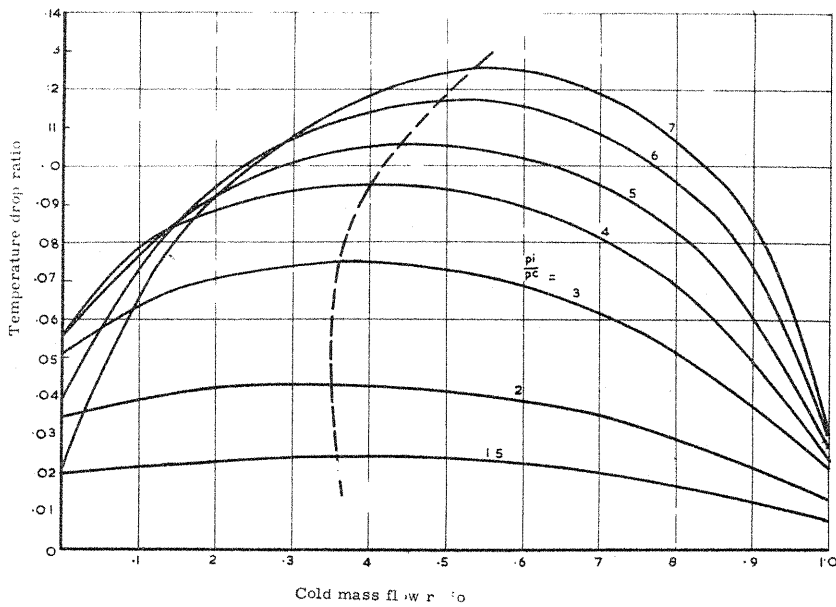


FIG. 17. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = 0.64$ ,  $\frac{d_h}{D} = 0.266$



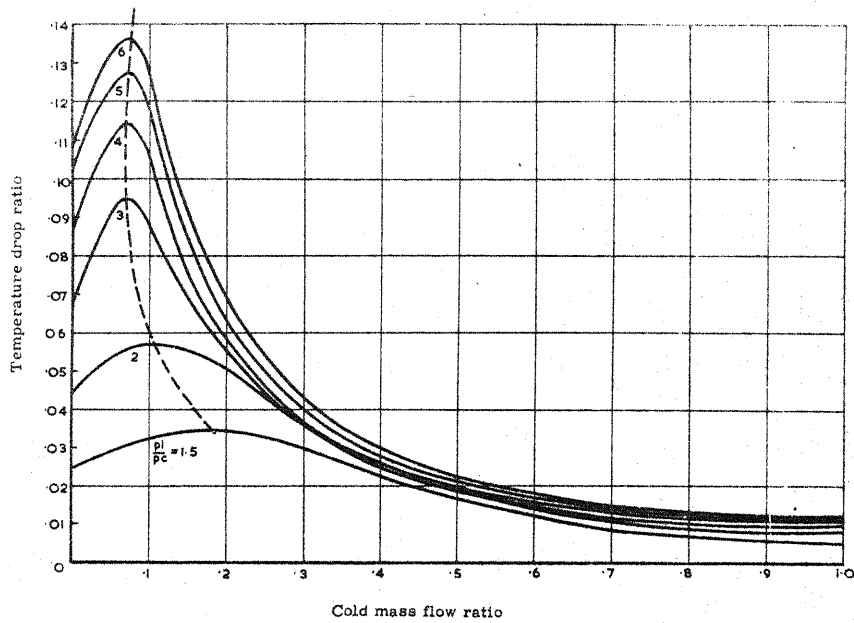


FIG. 18. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .167, \frac{d_1}{D} = .376$

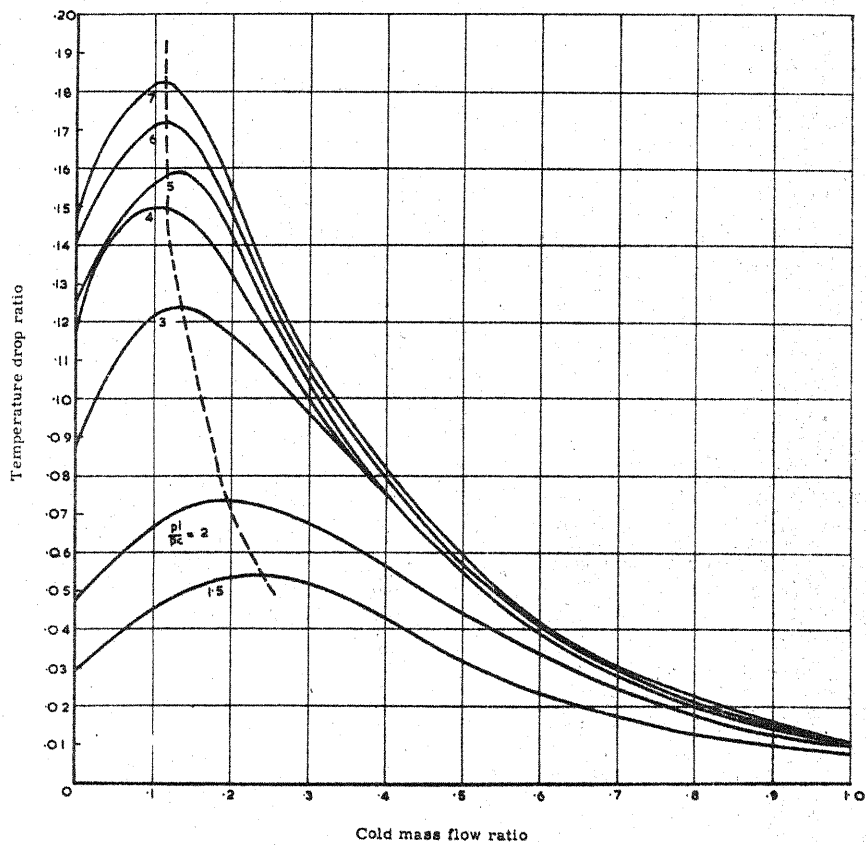


FIG. 19. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .250; \frac{d_1}{D} = .316$

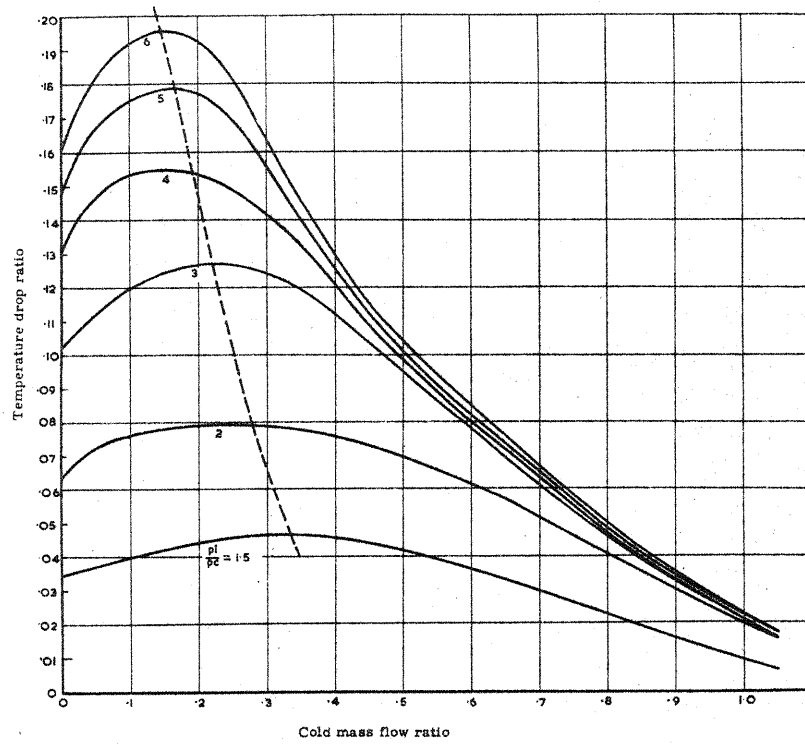


FIG. 20. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .333; \frac{d_i}{D} = .376$

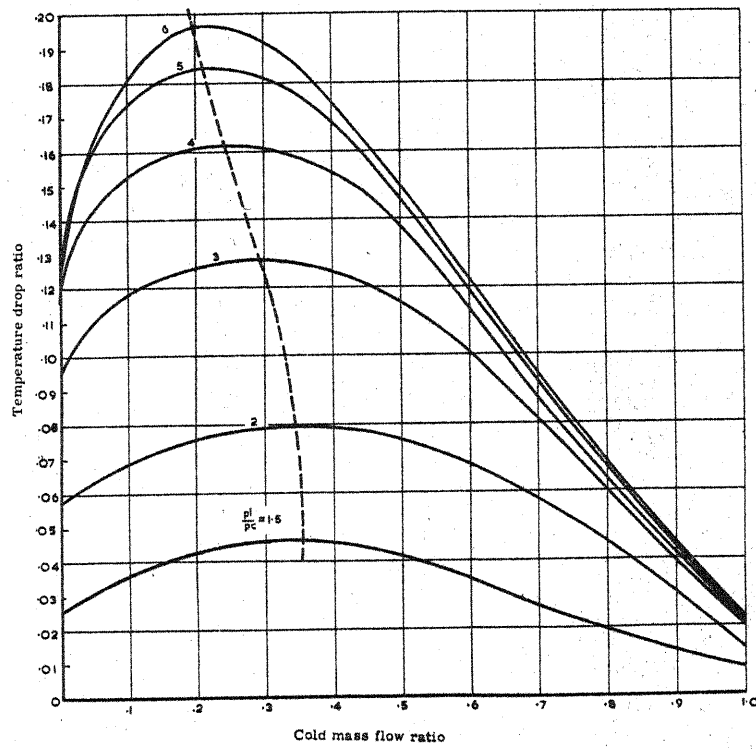


FIG. 21. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .417; \frac{d_i}{D} = .376$

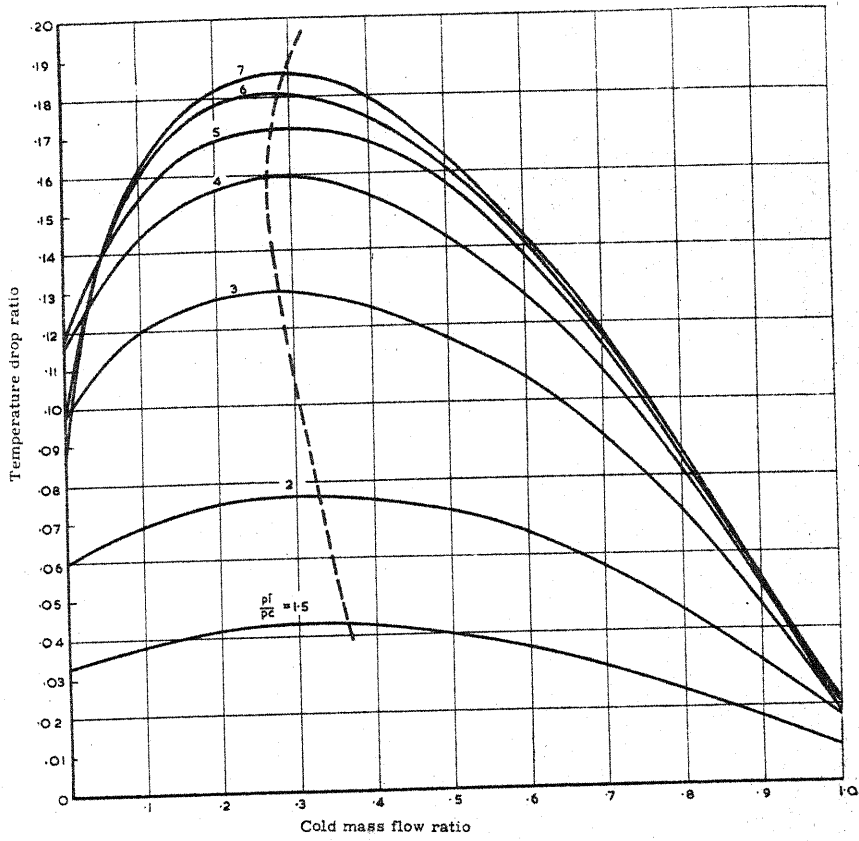


FIG. 22 VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .530; \frac{d_1}{D} = .376$

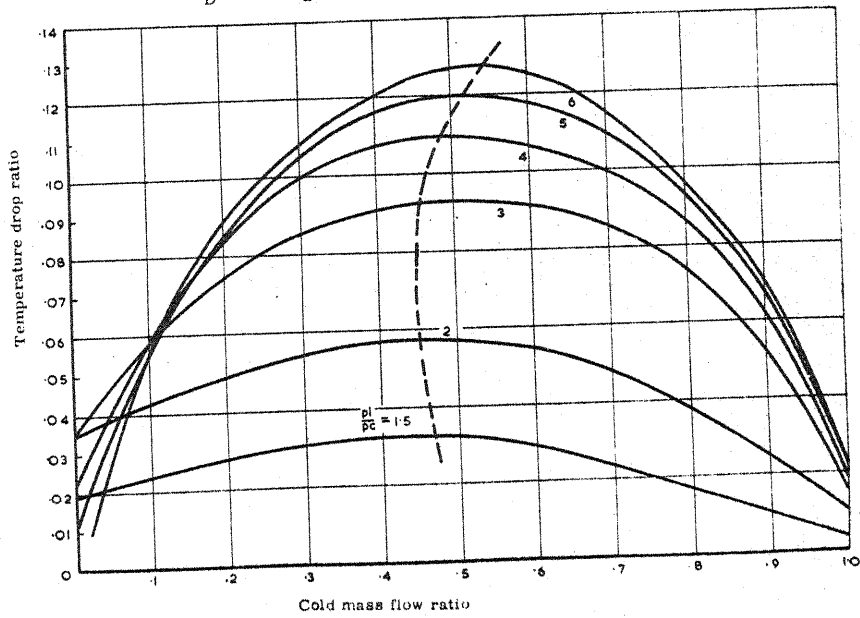


FIG. 23 VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .667; \frac{d_1}{D} = .376$

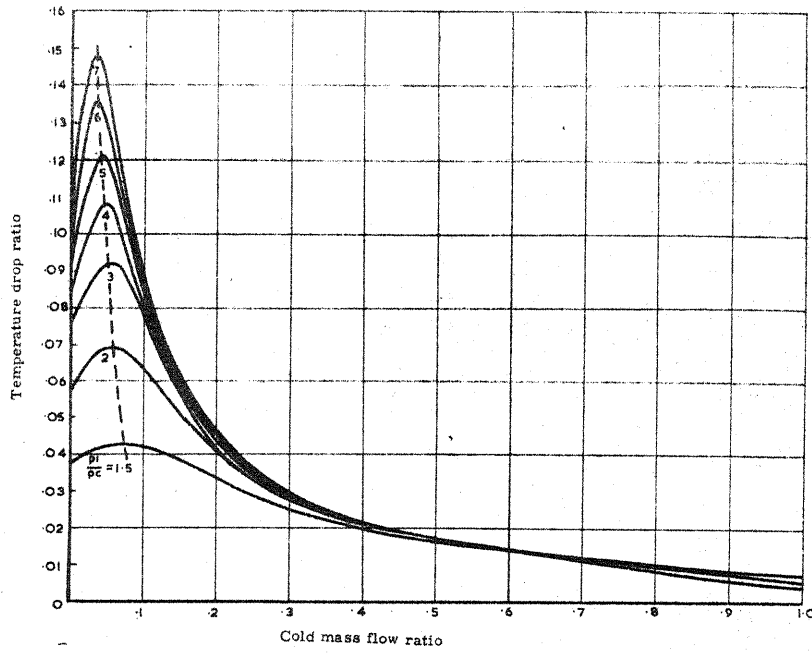


FIG. 24. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .167$ ;  $\frac{d_1}{D} = .461$

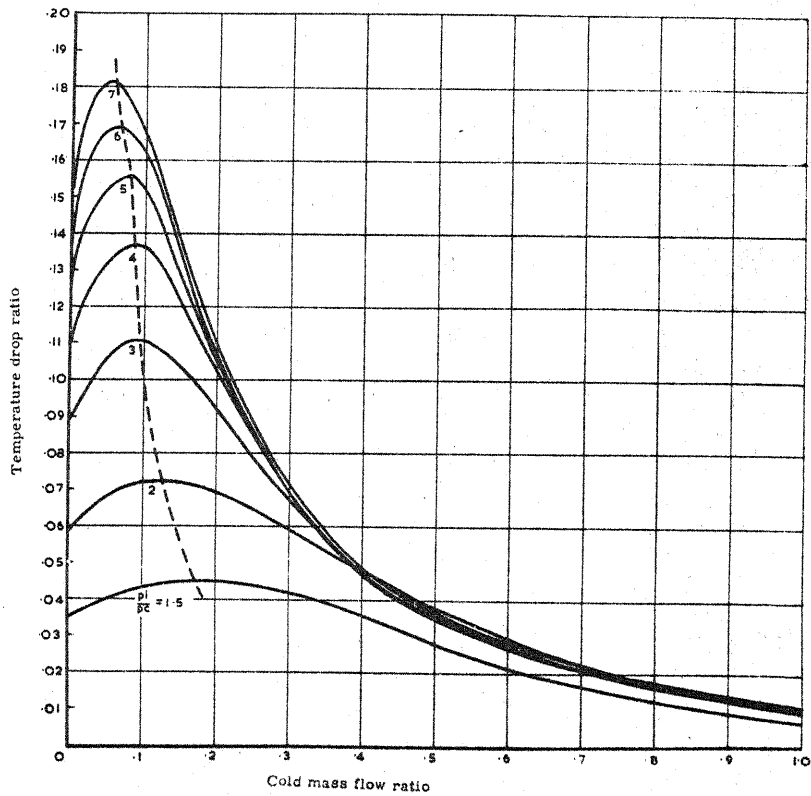


FIG. 25. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .20$ ;  $\frac{d_1}{D} = .461$

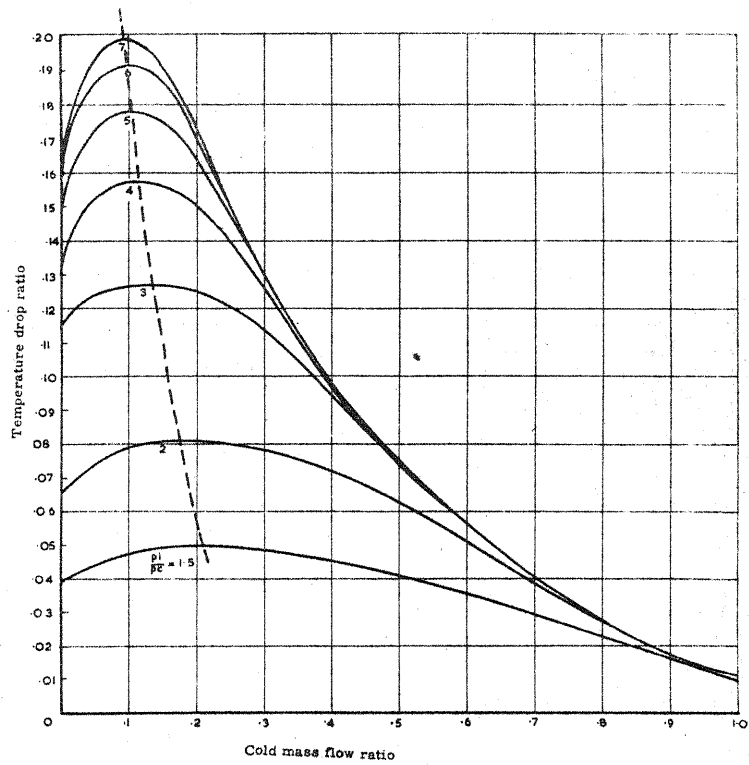


FIG. 26. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .333; \frac{d_1}{D} = .461$

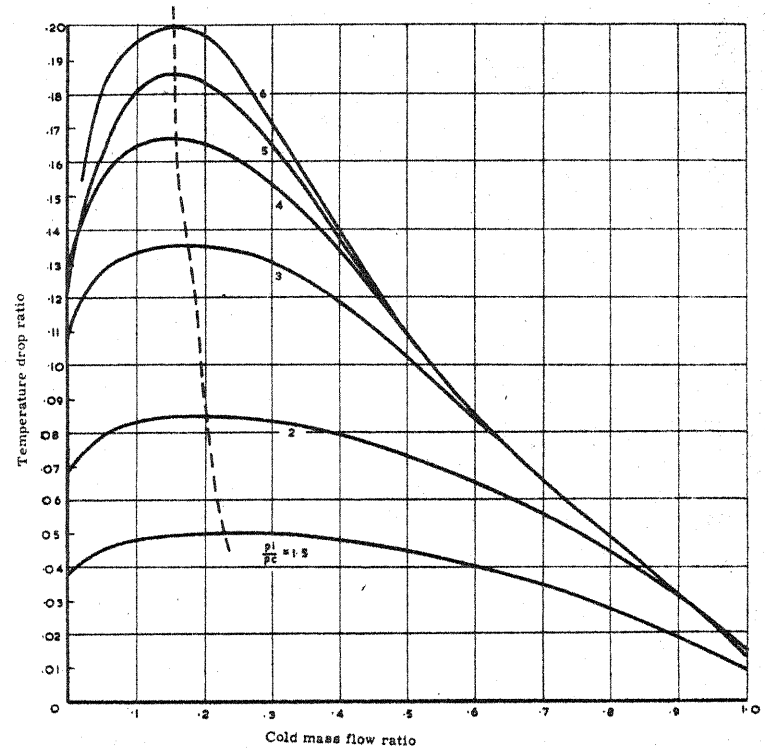


FIG. 27. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .417; \frac{d_1}{D} = .461$

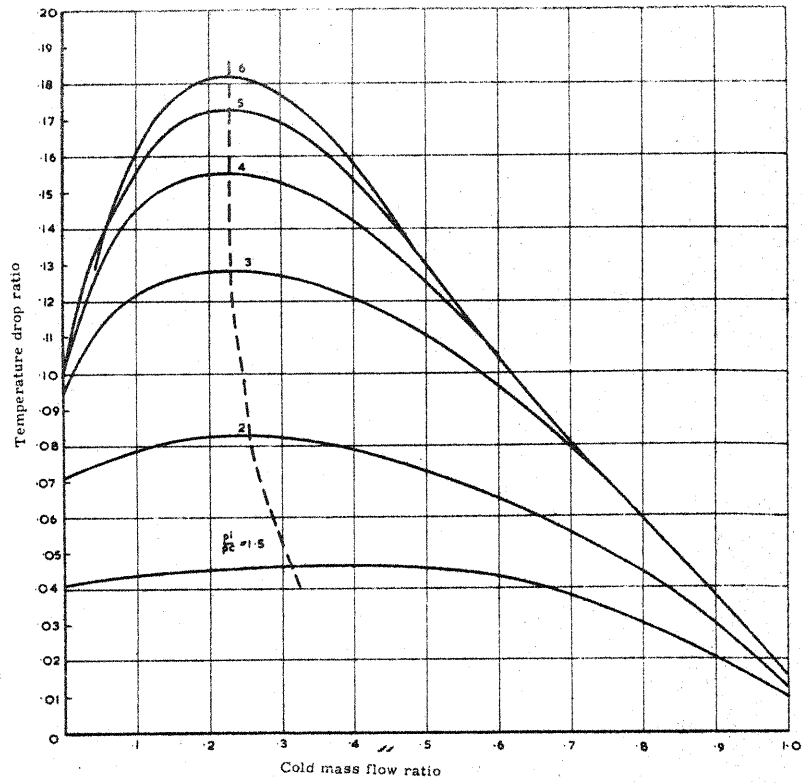


FIG. 28. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .50$ ;  $\frac{d_1}{D} = .461$

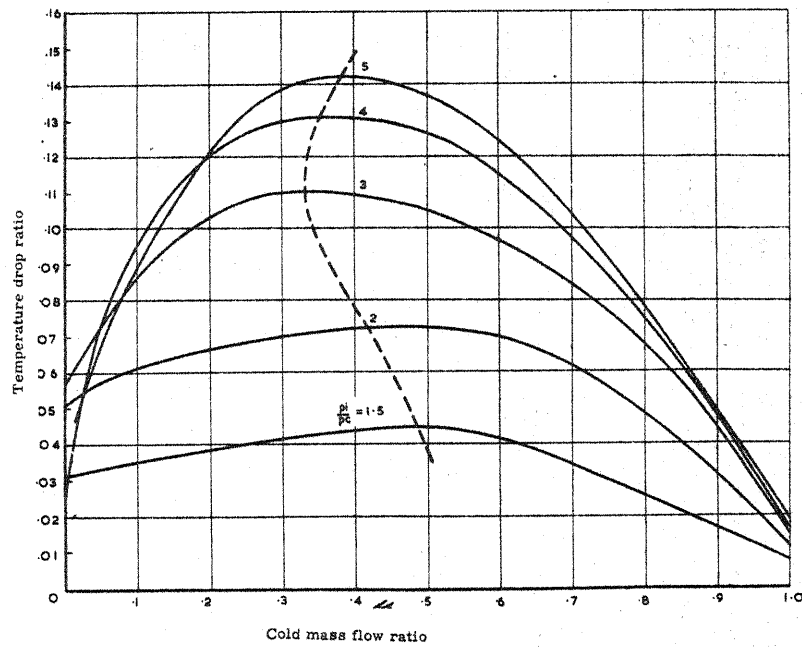


FIG. 29. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .583$ ;  $\frac{d_1}{D} = .461$

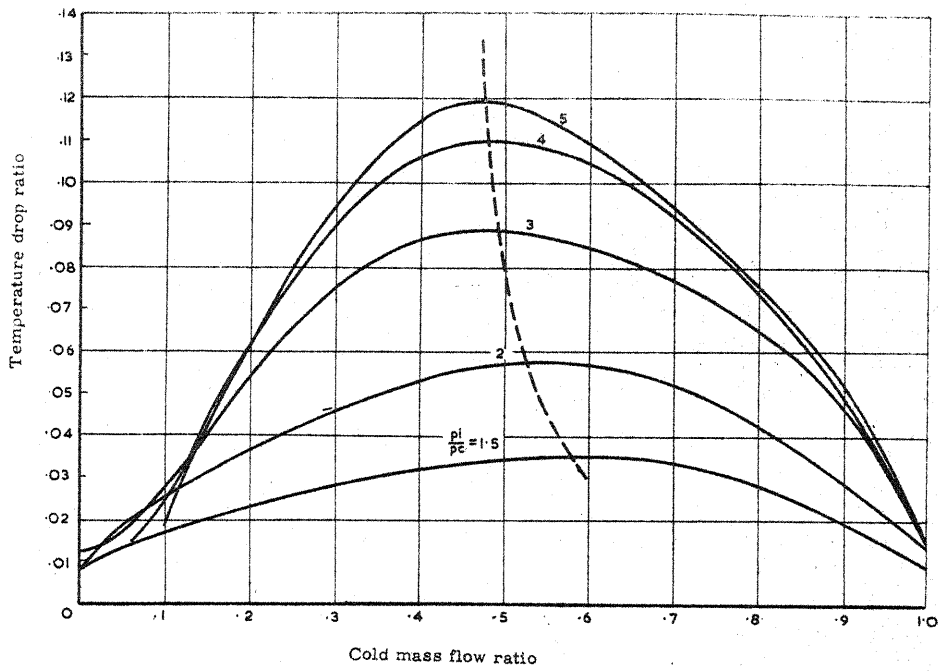


FIG. 30. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.

$$\frac{d_c}{D} = .667, \quad \frac{d_1}{D} = .461$$

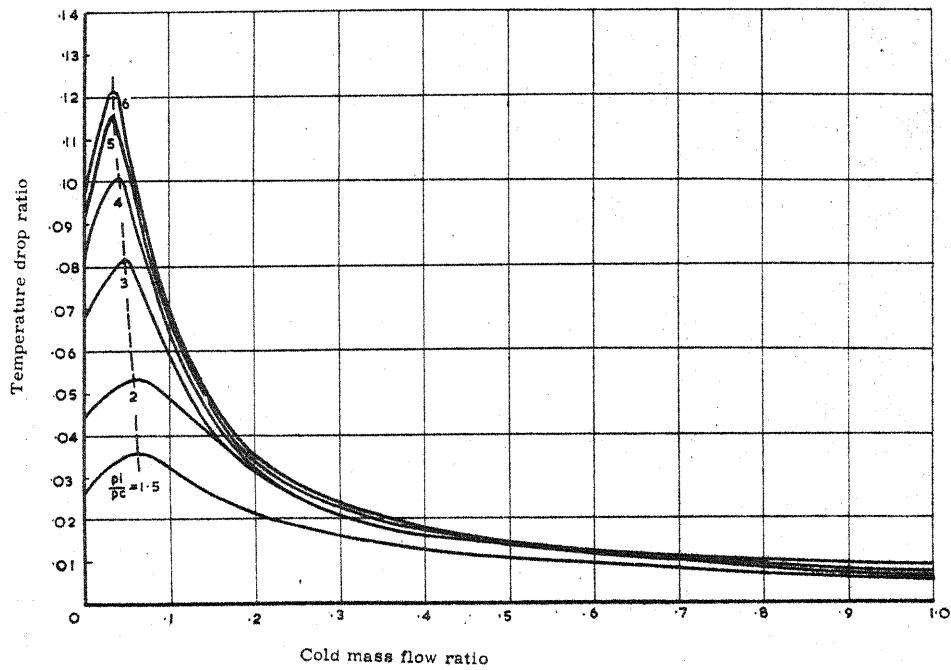


FIG. 31. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS

$$\frac{d_c}{D} = .167; \quad \frac{d_1}{D} = .532$$

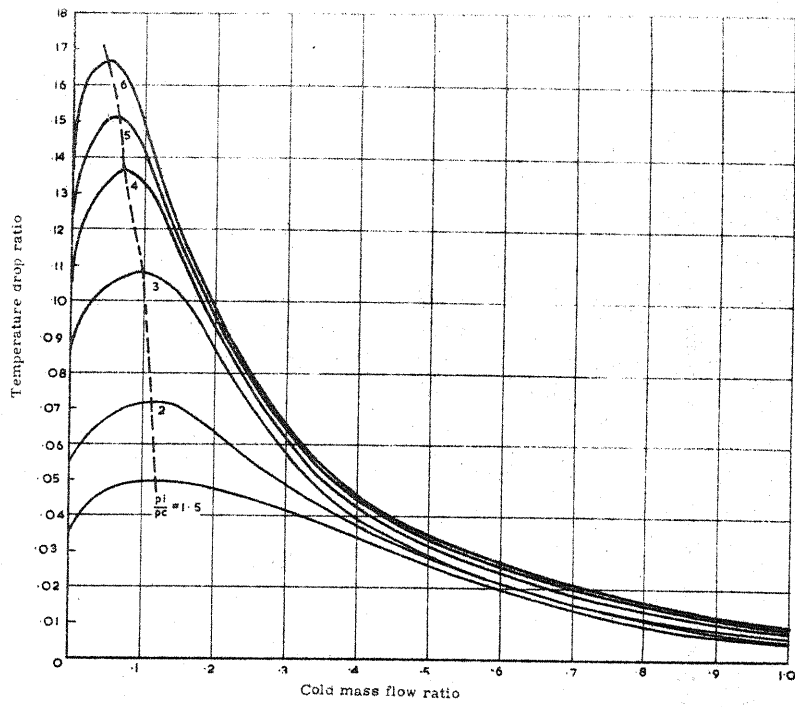


FIG. 32. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .250; \frac{d_i}{D} = .532$

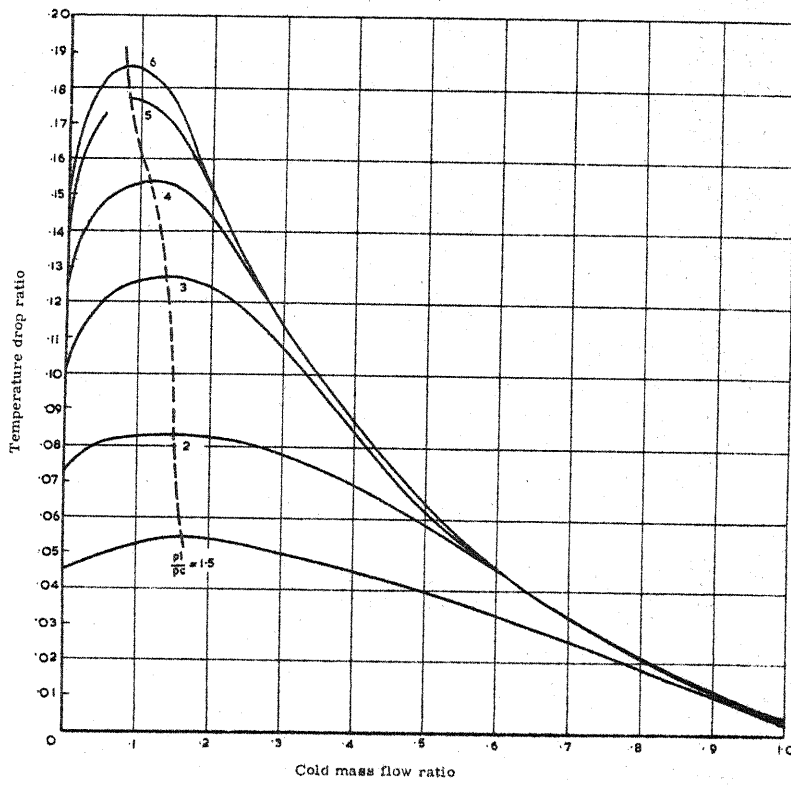


FIG. 33. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .333; \frac{d_i}{D} = .532$



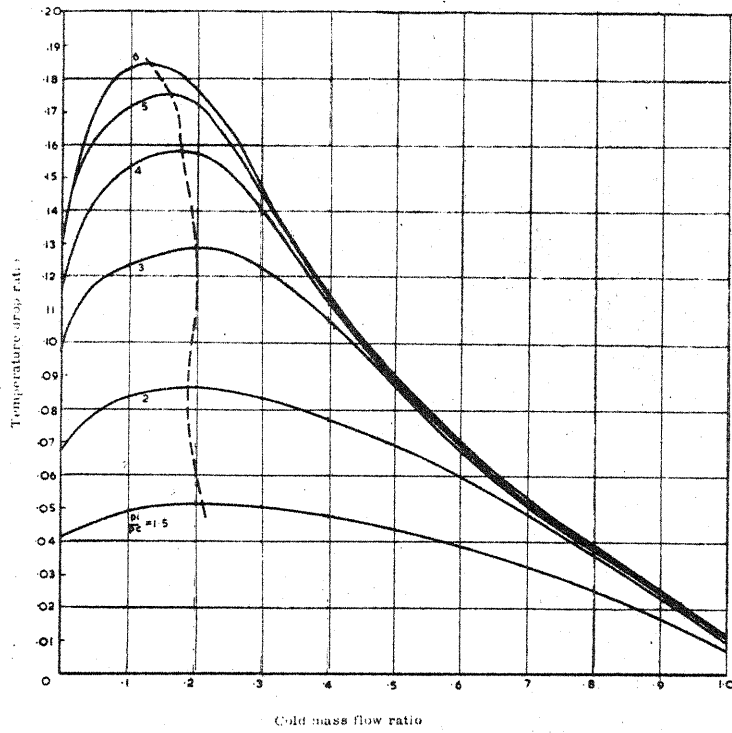


FIG. 34. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .417$ ;  $\frac{d_i}{D} = .332$   
 $\frac{p_1}{p_2} = 1.5$

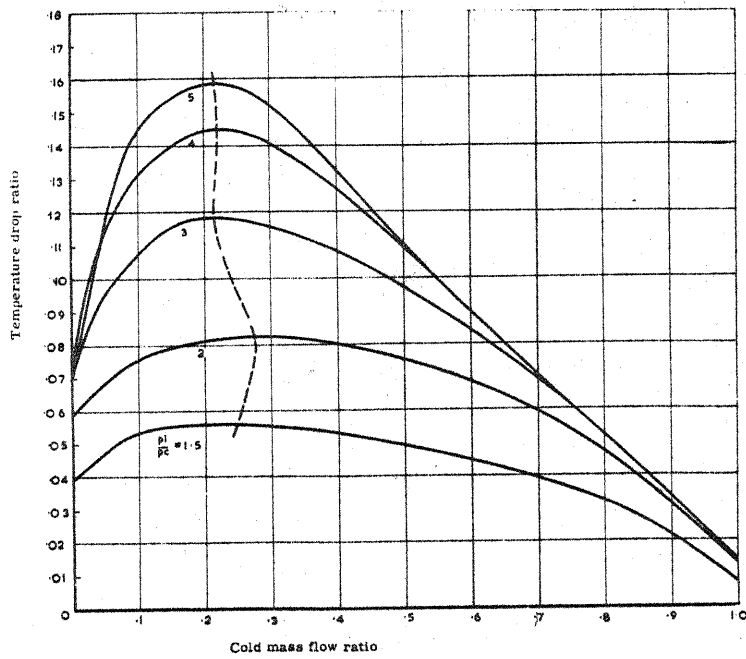


FIG. 35. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .500$ ;  $\frac{d_i}{D} = .532$   
 $\frac{p_1}{p_2} = 1.5$

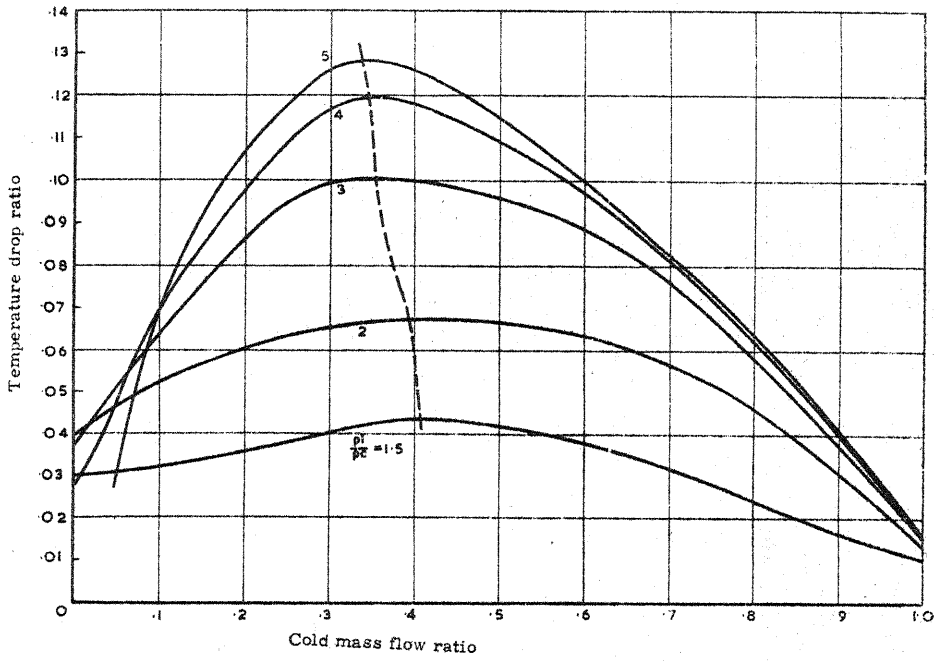


FIG. 36. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .583$ ;  $\frac{d_1}{D} = .532$

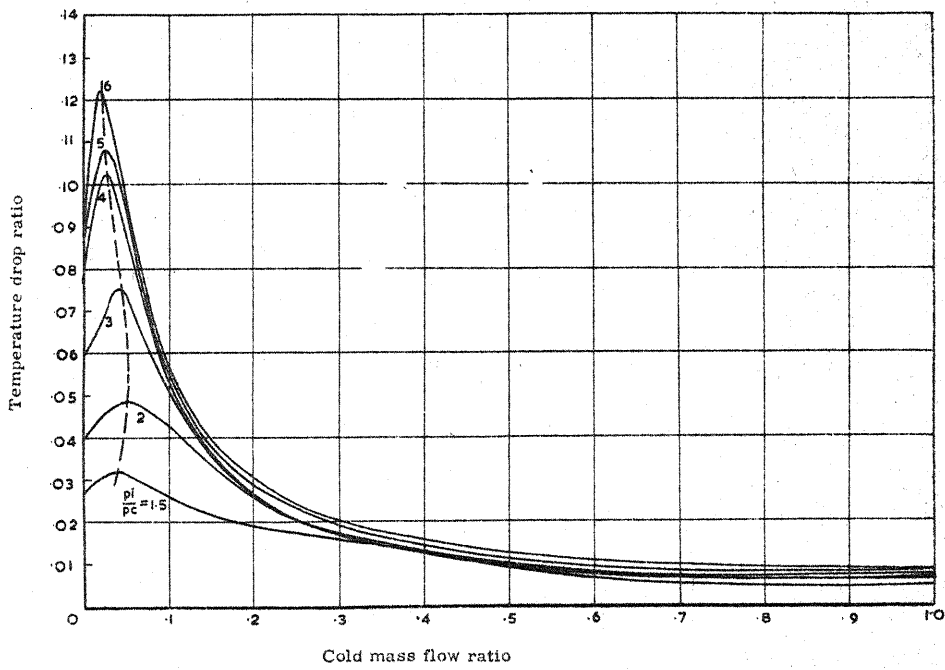


FIG. 37. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .167$ ;  $\frac{d_1}{D} = .595$

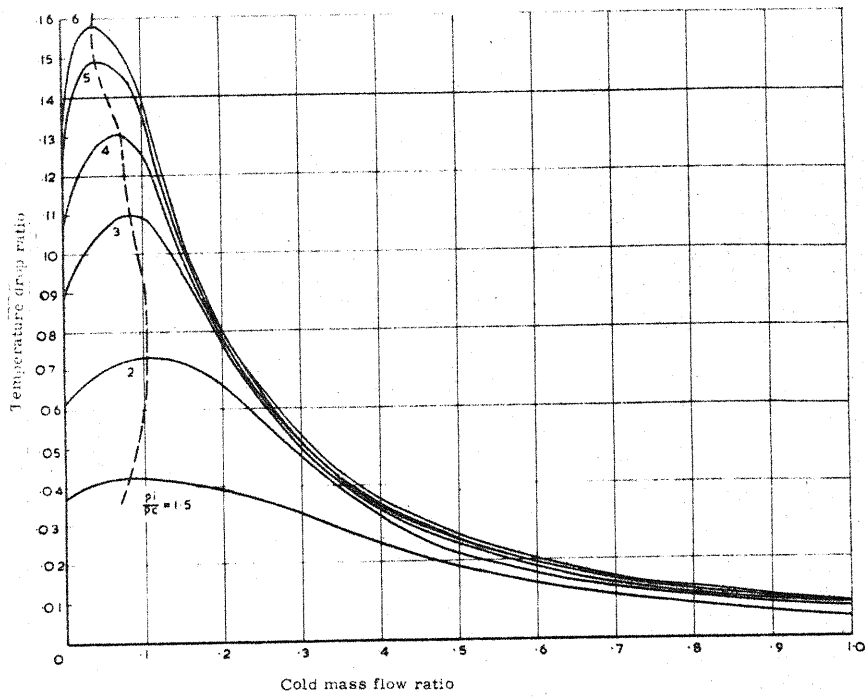


FIG. 38. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .250; \frac{d_i}{D} = .595$

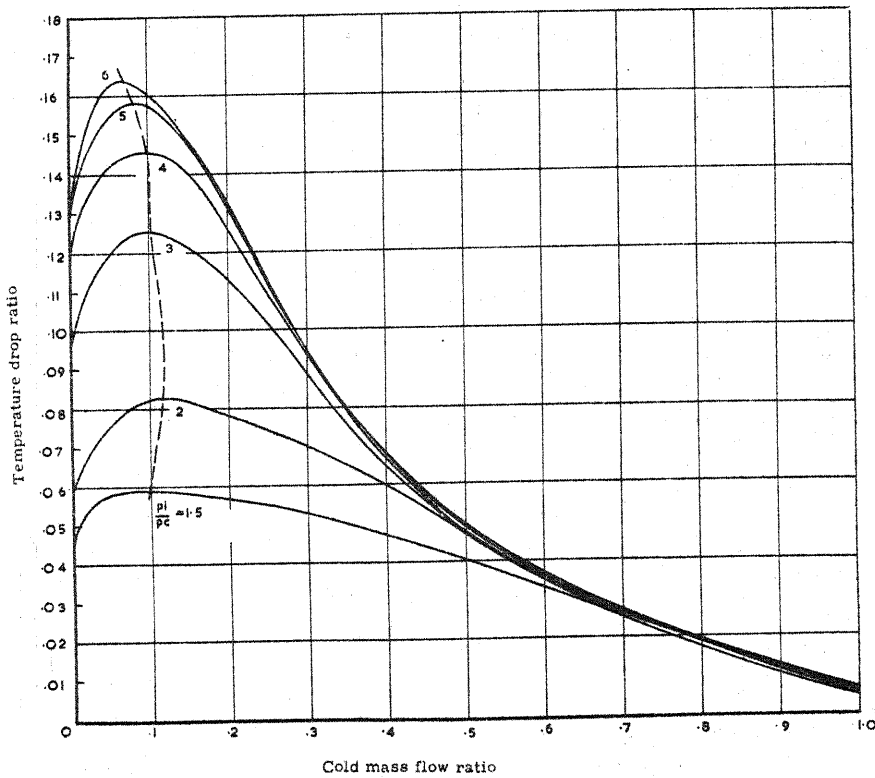


FIG. 39. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .333; \frac{d_i}{D} = .595$

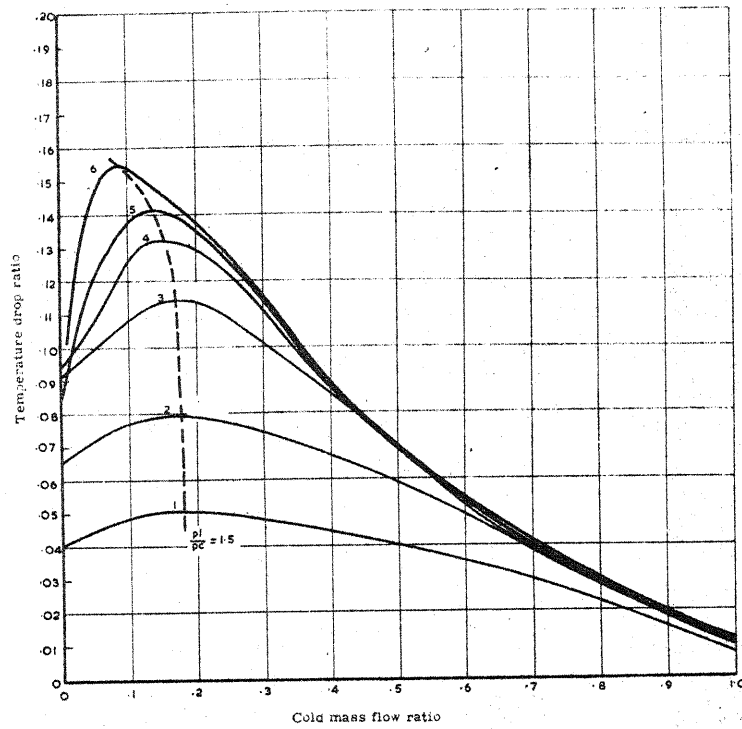


FIG. 40. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_c}{D} = .417, \frac{d_1}{D} = .595$

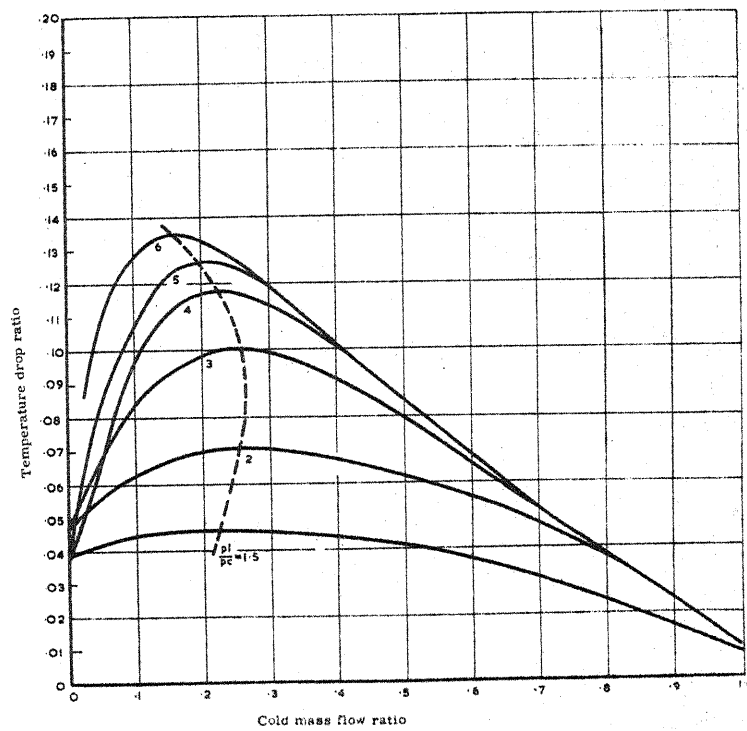


FIG. 41. VARIATION OF TEMPERATURE DROP RATIO WITH COLD MASS FLOW RATIO FOR VARIOUS PRESSURE RATIOS  
 $\frac{d_c}{D} = .500, \frac{d_1}{D} = .595$

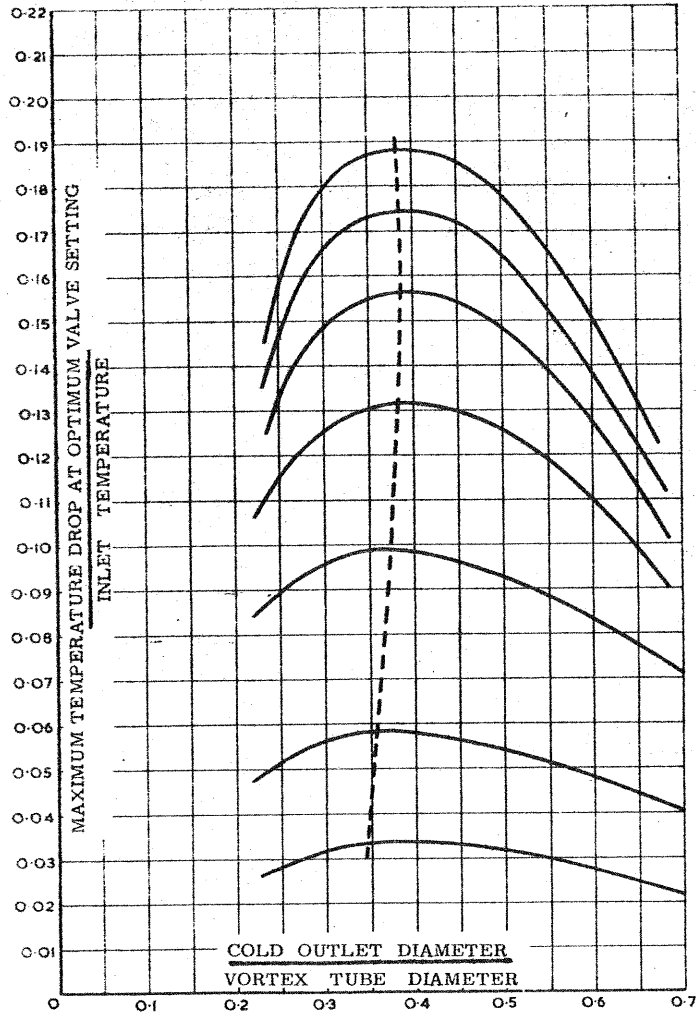


FIG. 42 VARIATION OF MAXIMUM TEMPERATURE DROP RATIO WITH COLD OUTLET DIAMETER RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_1}{D} = .266$

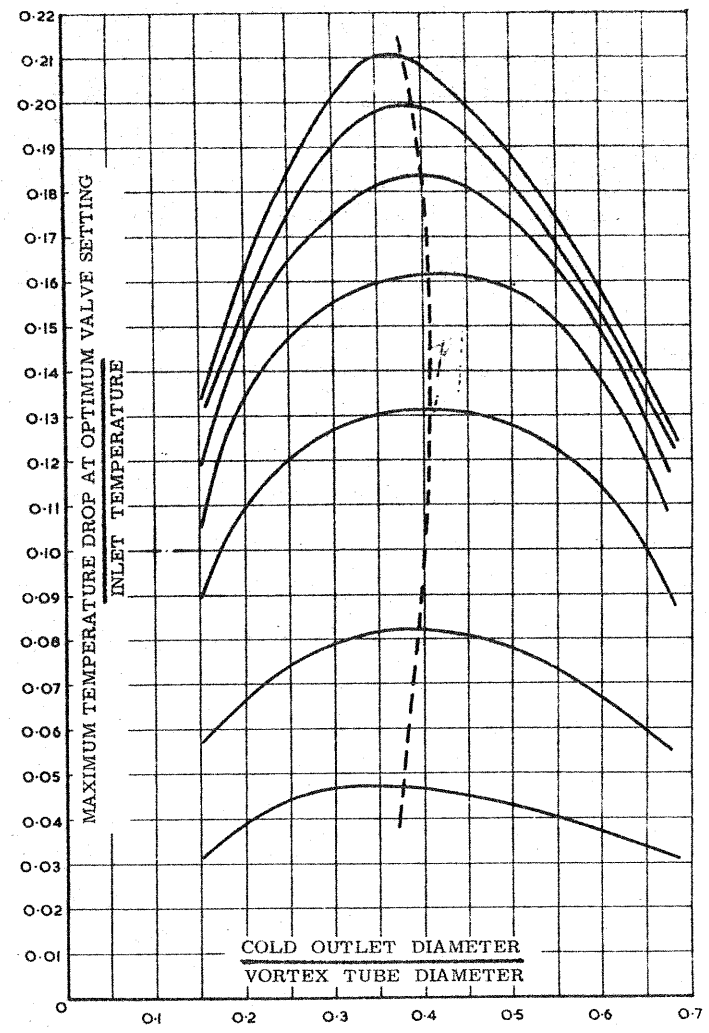


FIG. 43. VARIATION OF MAXIMUM TEMPERATURE DROP RATIO WITH COLD OUTLET DIAMETER RATIO FOR VARIOUS PRESSURE RATIOS.  
 $\frac{d_1}{D} = .376$

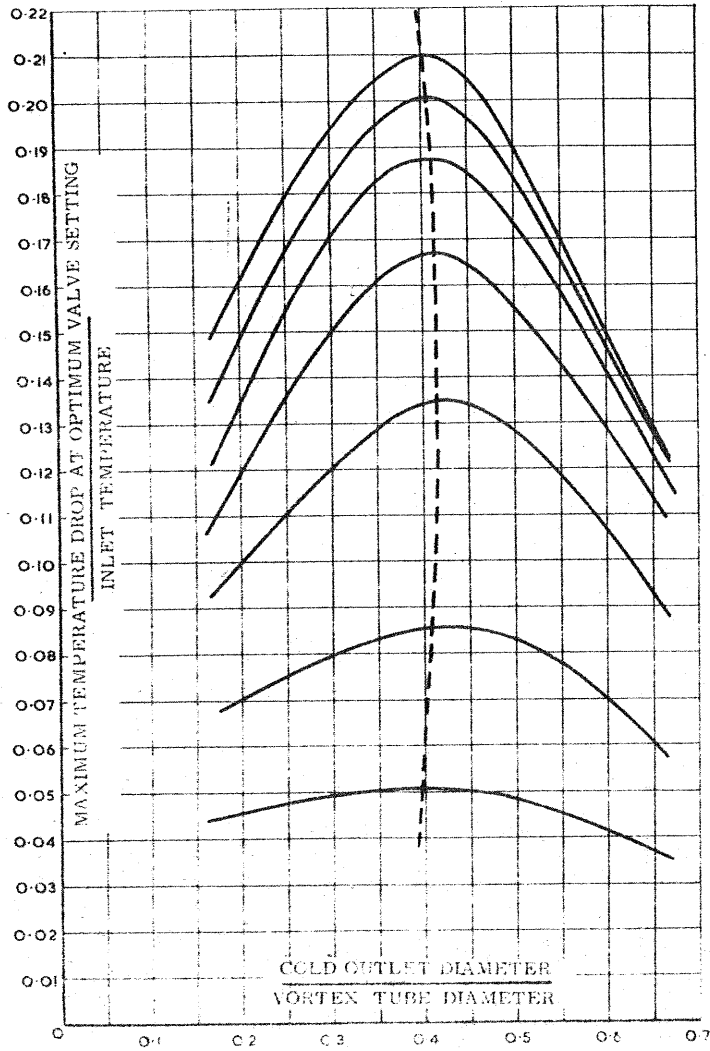


FIG. 44. VARIATION OF MAXIMUM TEMPERATURE DROP RATIO WITH COLD OUTLET DIAMETER RATIO FOR VARIOUS PRESSURE RATIOS.

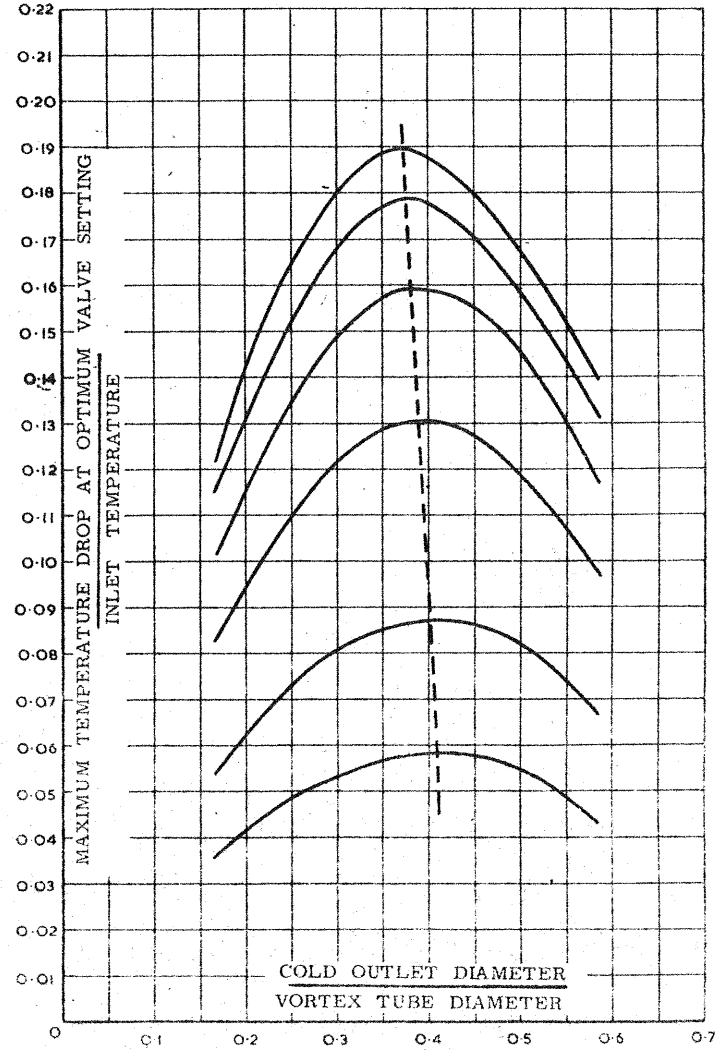


FIG. 45. VARIATION OF MAXIMUM TEMPERATURE DROP RATIO WITH COLD OUTLET DIAMETER RATIO FOR VARIOUS PRESSURE RATIOS.

d = 0.333

B

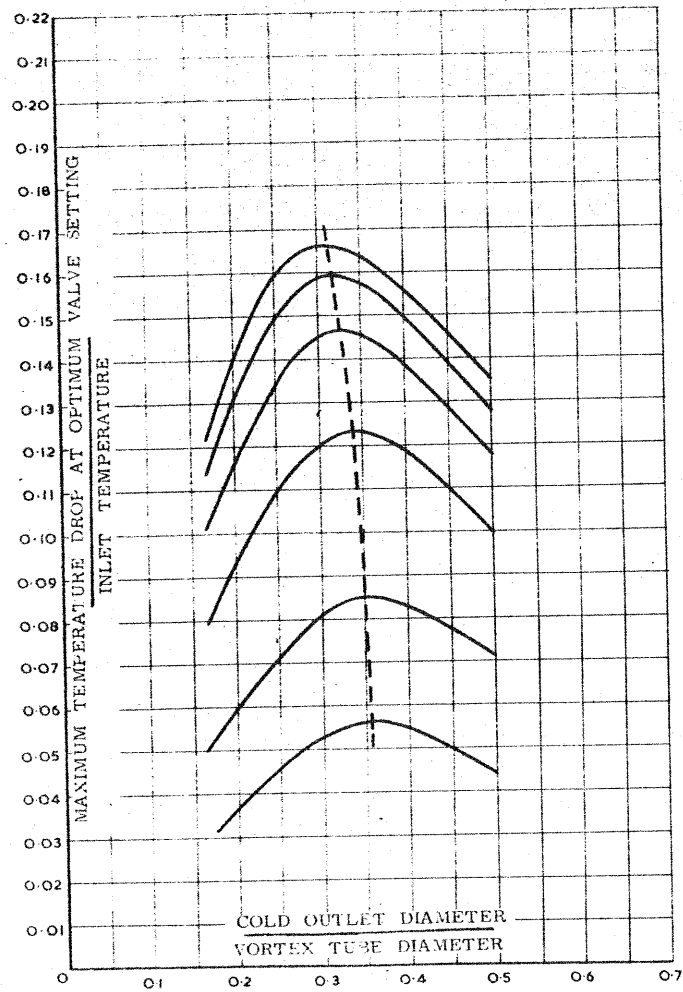


FIG. 46. VARIATION OF MAXIMUM TEMPERATURE DROP RATIO WITH COLD OUTLET DIAMETER RATIO FOR VARIOUS PRESSURE RATIOS

$$\frac{d_1}{D} = .595$$

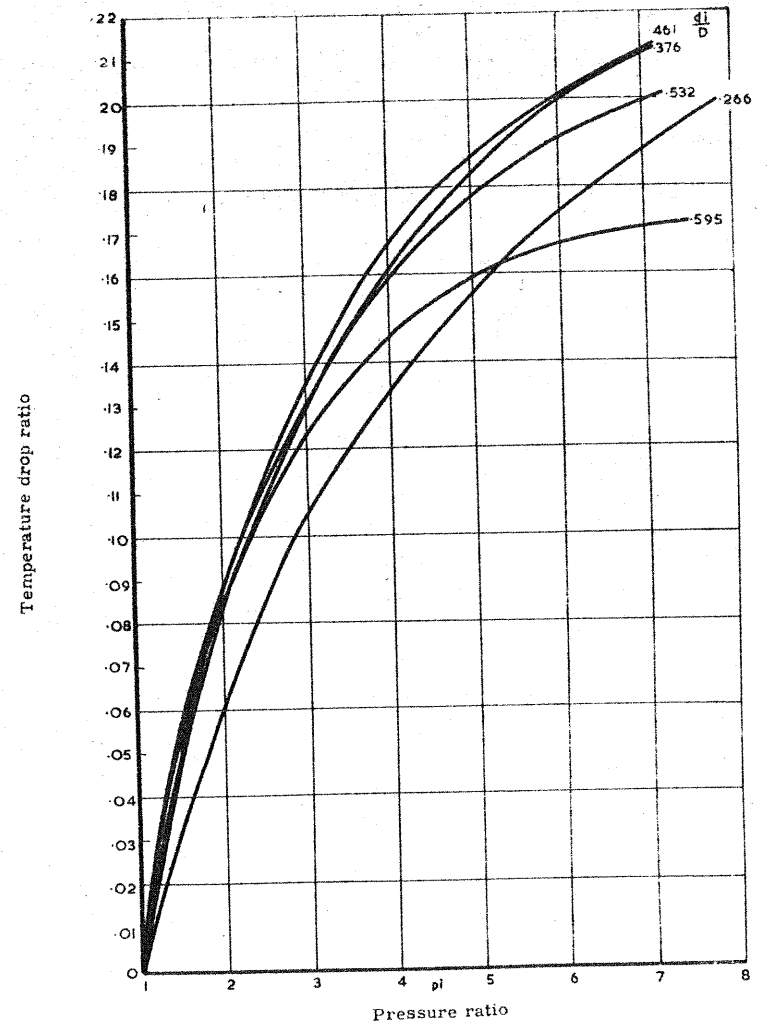


Fig. 47. VARIATION OF TEMPERATURE DROP RATIO WITH PRESSURE RATIO FOR OPTIMUM COLD OUTLET DIAMETER, OPTIMUM VALUE SETTING AND GIVEN INLET SIZE  $\frac{d_1}{D}$

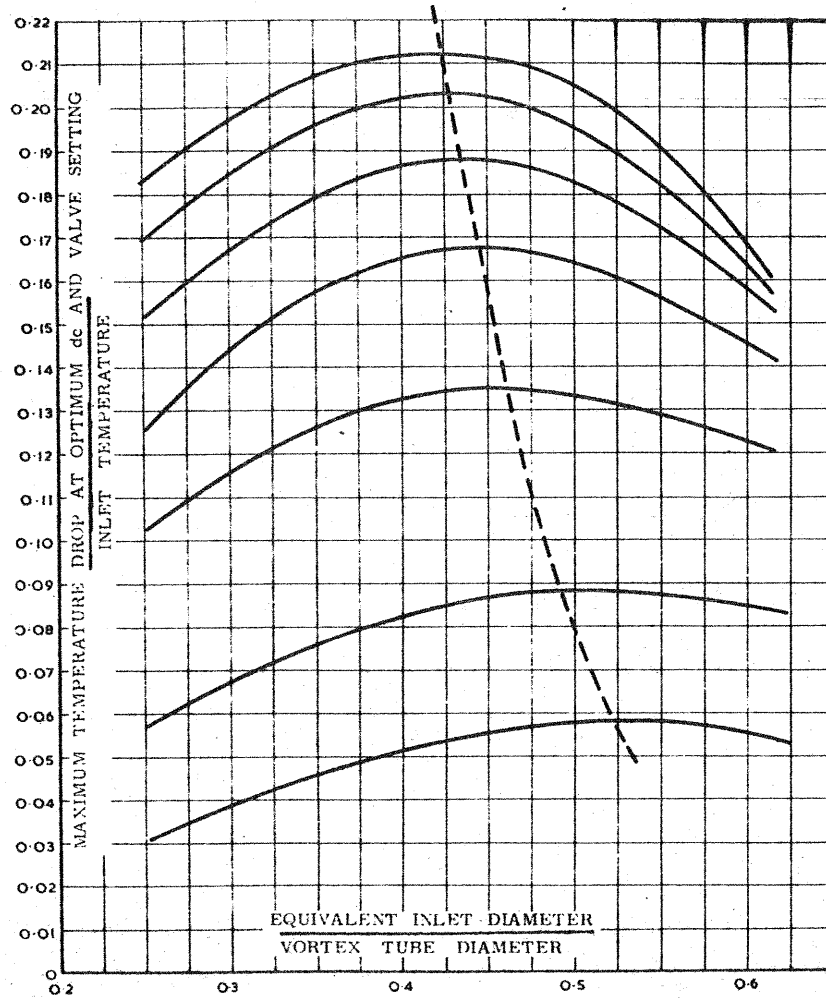


FIG. 48. VARIATION OF TEMPERATURE DROP RATIO WITH INLET SIZE FOR OPTIMUM COLD OUTLET AND VALVE SETTING

$$\frac{P_1}{P_c} = 1.5, 2.0, 3.0, 4.0, 5.0, 6.0, 7.0$$

$$\left(\frac{\Delta T_c}{T_1}\right)_2 \sim \frac{d_i}{D}$$

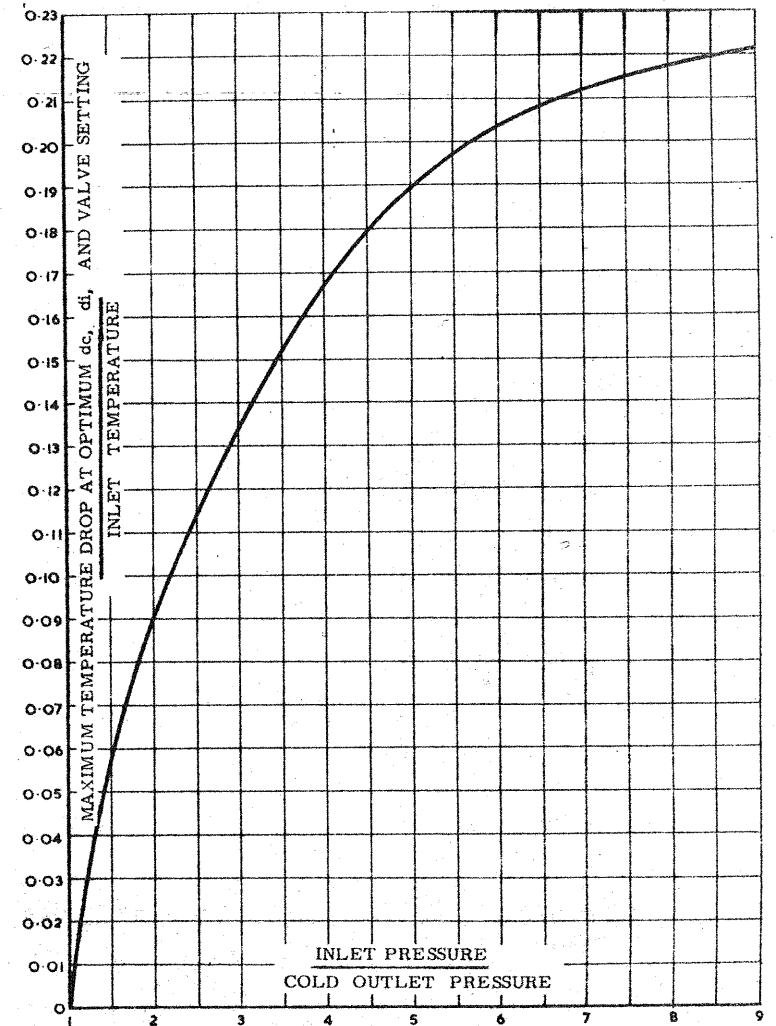


FIG. 49. VARIATION OF TEMPERATURE DROP EFFICIENCY WITH PRESSURE RATIO AT OPTIMUM COLD OUTLET SIZE, OPTIMUM VALVE SETTING AND FIXED INLET DIAMETER.

$$\frac{d_i}{D} = .266, .376, .461, .532, .595$$

$$\frac{\Delta T_{c2}}{\Delta T_{c \text{ ISEN}}} \sim \frac{P_1}{P_c}$$



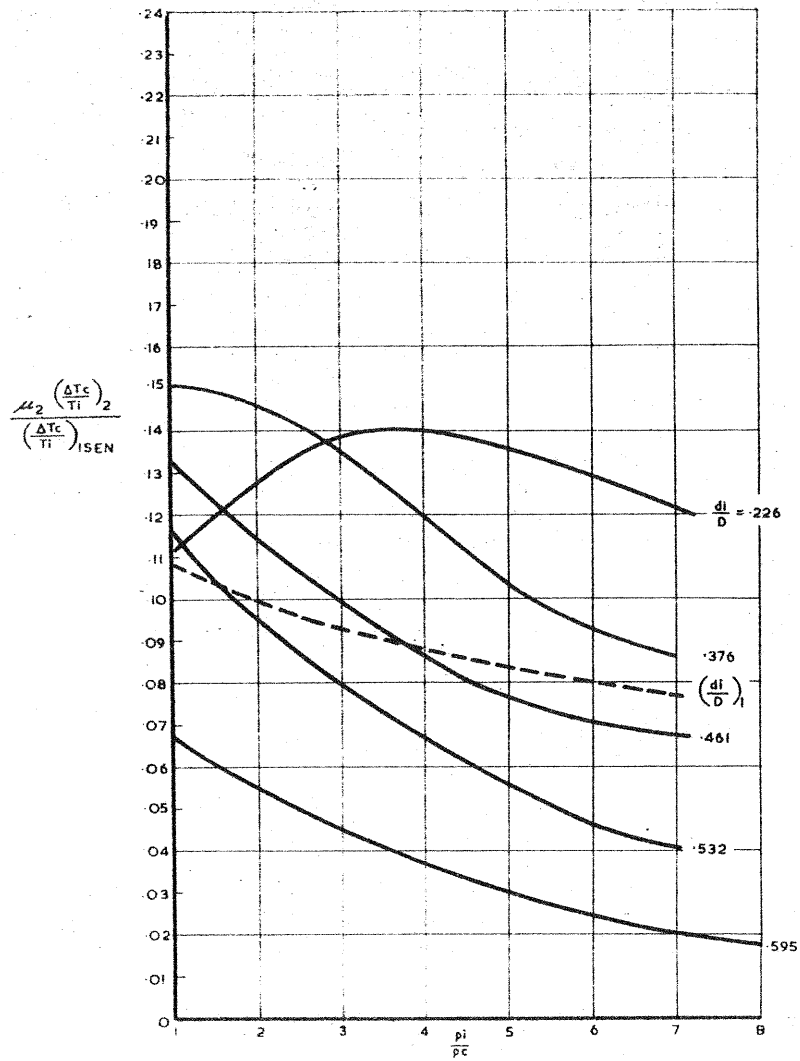


FIG. 5. VARIATION OF REFRIGERATION EFFICIENCY WITH PRESSURE RATIO AT OPTIMUM VALVE SETTING AND COLD OUTLET DIAMETER FOR MAXIMUM TEMPERATURE DROP RATIO.

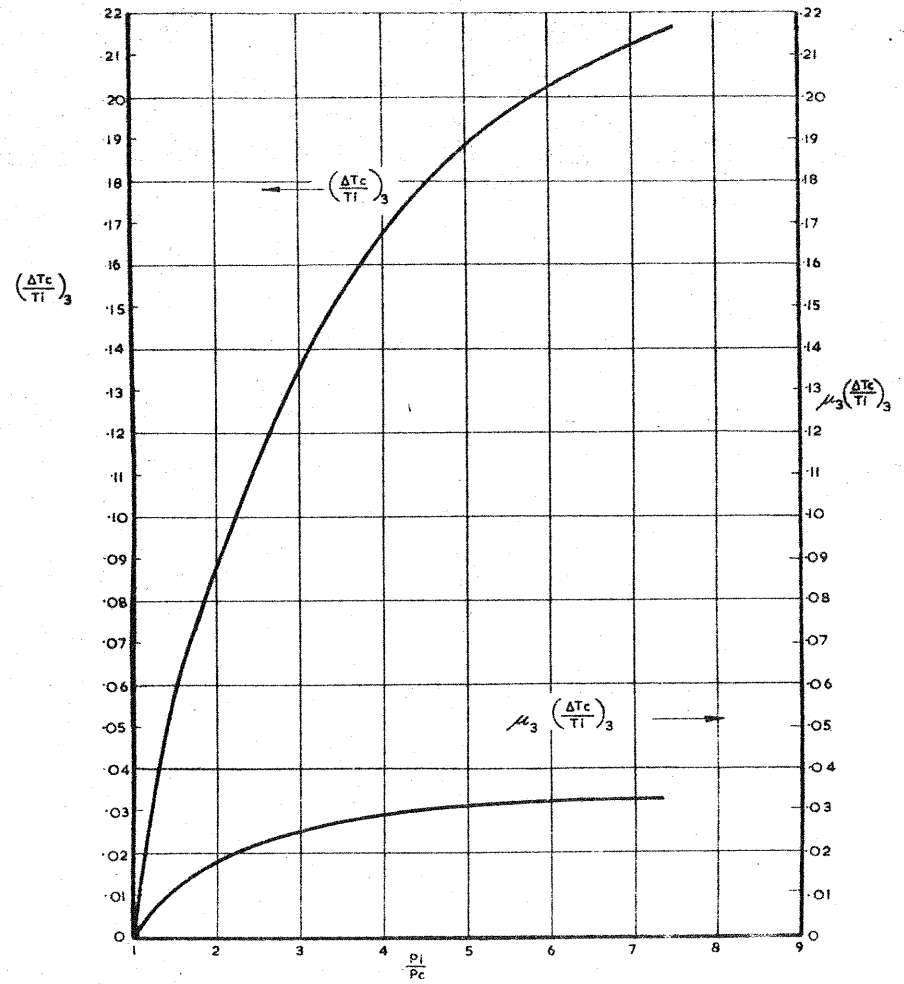


FIG. 51. VARIATION OF TEMPERATURE DROP RATIO AND REFRIGERATION RATIO WITH PRESSURE RATIO AT MAXIMUM TEMPERATURE DROP AND OPTIMUM CONDITIONS.

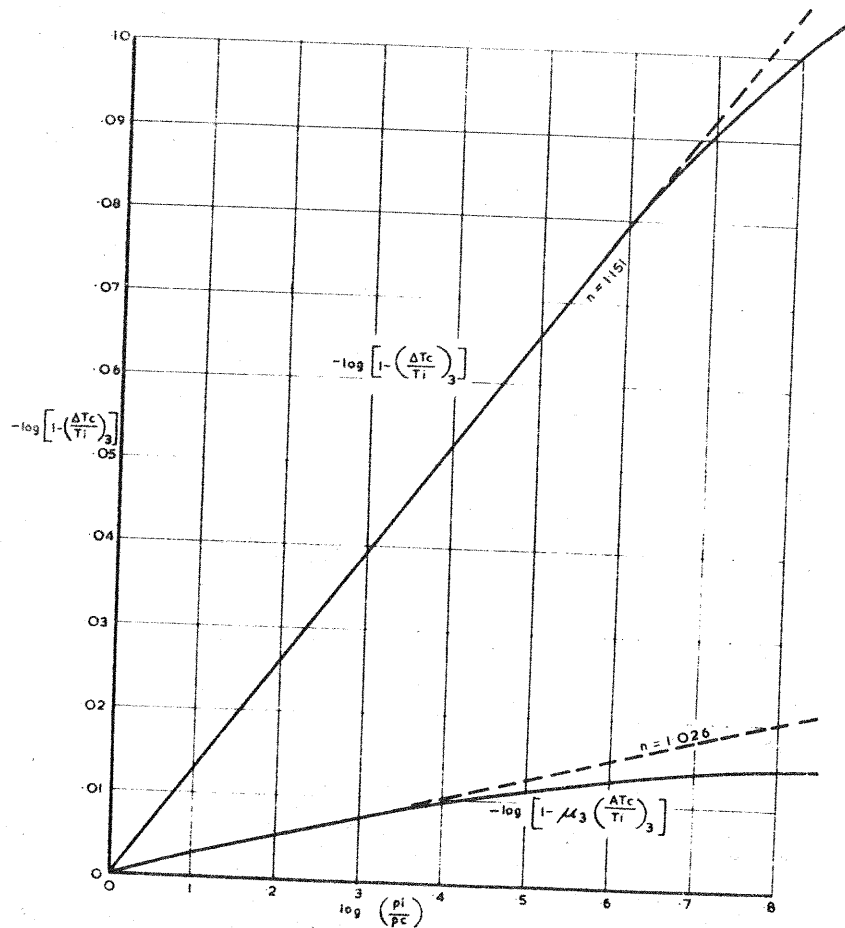


FIG. 32. LOWER LAW VARIATION OF TEMPERATURE DROP RATIO AND REFRIGERATION RATE WITH PRESSURE RATIO AT MAXIMUM TEMPERATURE DROP.

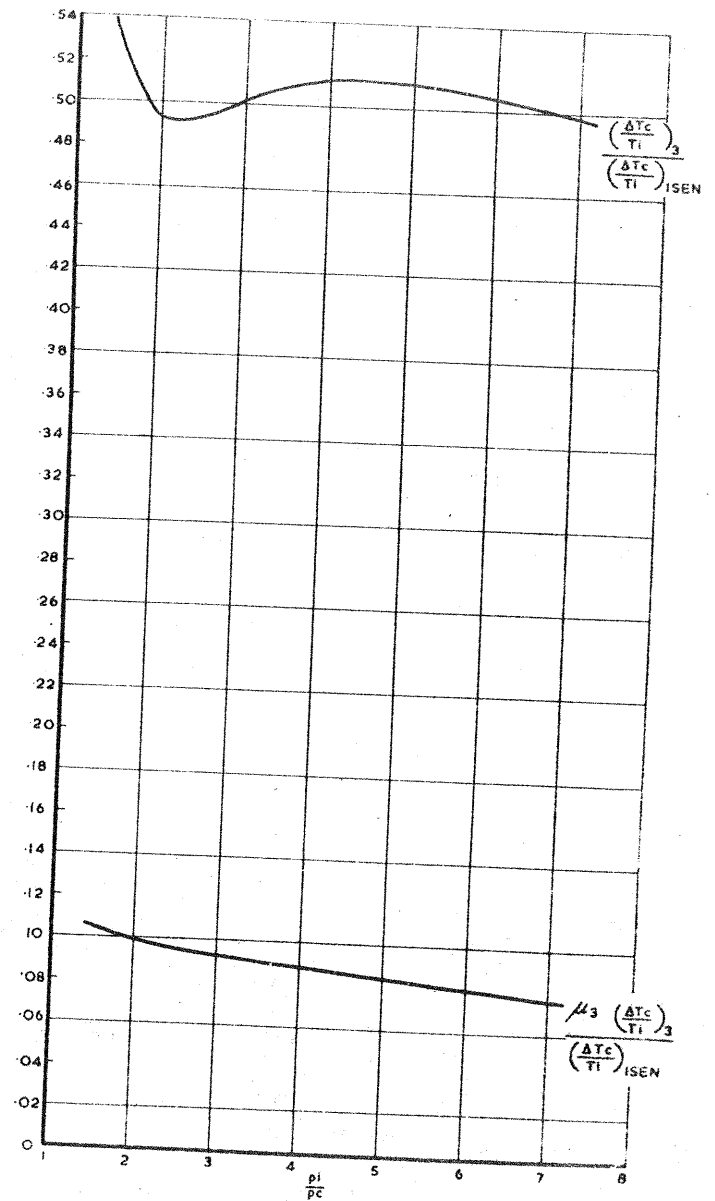


FIG. 33. VARIATION OF TEMPERATURE DROP EFFICIENCY AND REFRIGERATION EFFICIENCY WITH PRESSURE RATIO AT MAXIMUM TEMPERATURE DROP.

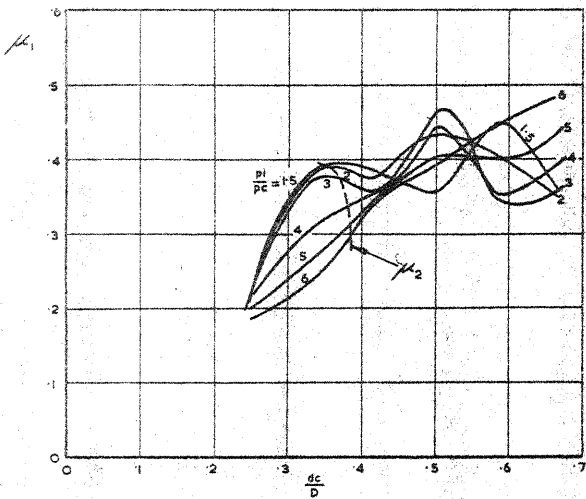


FIG. 54. VARIATION OF COLD MASS FLOW RATIO WITH COLD OUTLET DIAMETER RATIO AT OPTIMUM VALVE SETTING  
 $d_1/D = .268$

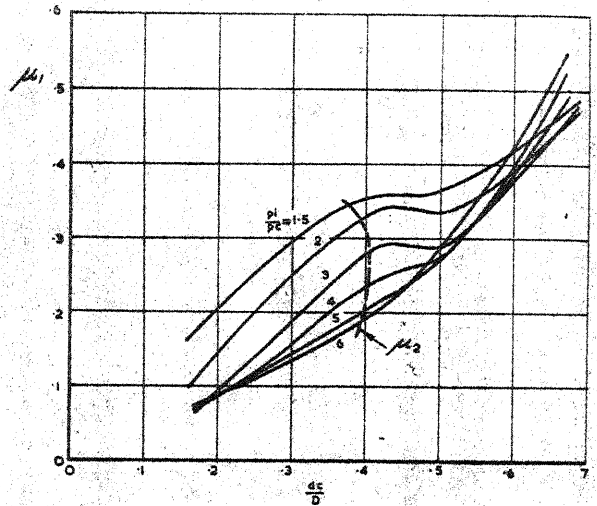


FIG. 55. VARIATION OF COLD MASS FLOW RATIO WITH COLD OUTLET DIAMETER RATIO AT OPTIMUM VALVE SETTING  
 $d_1/D = .376$

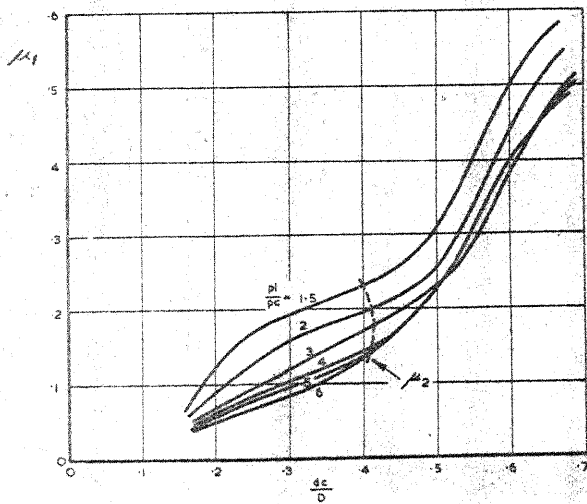


FIG. 56. VARIATION OF COLD MASS FLOW RATIO WITH COLD OUTLET DIAMETER RATIO AT OPTIMUM VALVE SETTING  
 $d_1/D = .401$

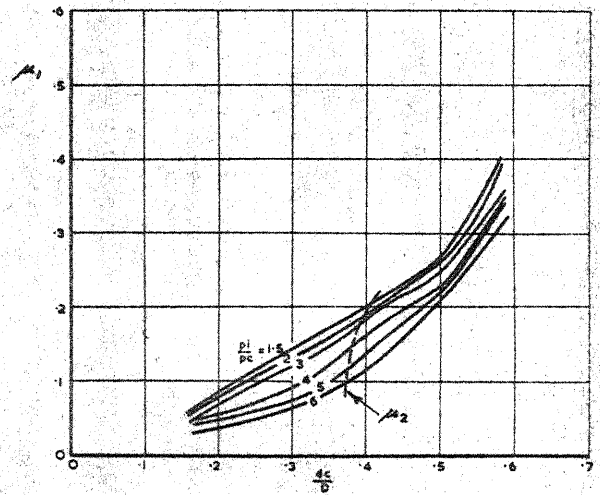


FIG. 57. VARIATION OF COLD MASS FLOW RATIO WITH COLD OUTLET DIAMETER RATIO AT OPTIMUM VALVE SETTING  
 $d_1/D = .532$

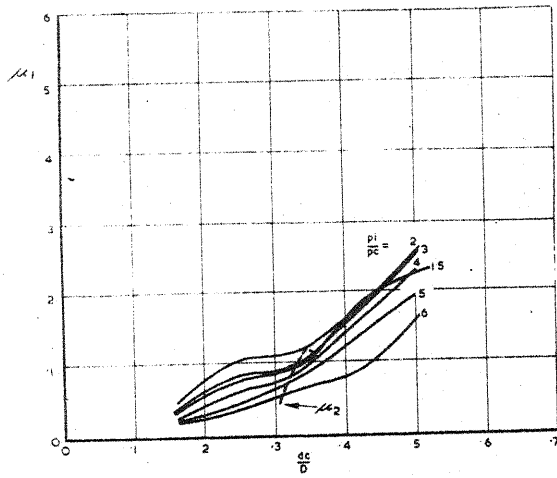


FIG. 58. VARIATION OF COLD MASS FLOW RATIO WITH COLD OUTLET DIAMETER RATIO AT OPTIMUM VALVE SETTING  
 $\frac{d_1}{D} = .595$

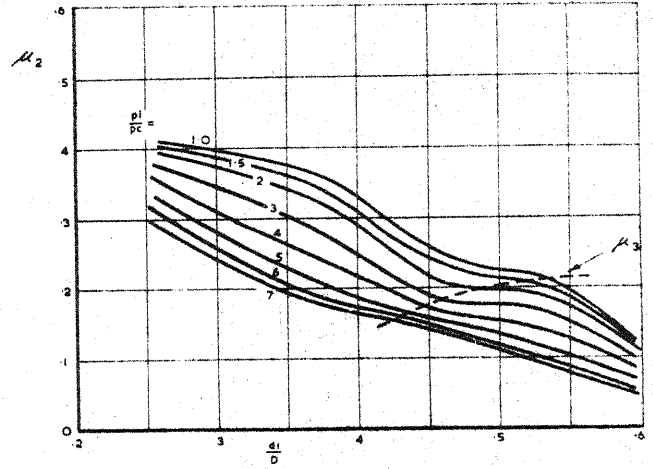


FIG. 59. VARIATION OF COLD MASS FLOW RATIO WITH INLET DIAMETER RATIO AT OPTIMUM VALVE SETTING AND OPTIMUM COLD OUTLET DIAMETER.

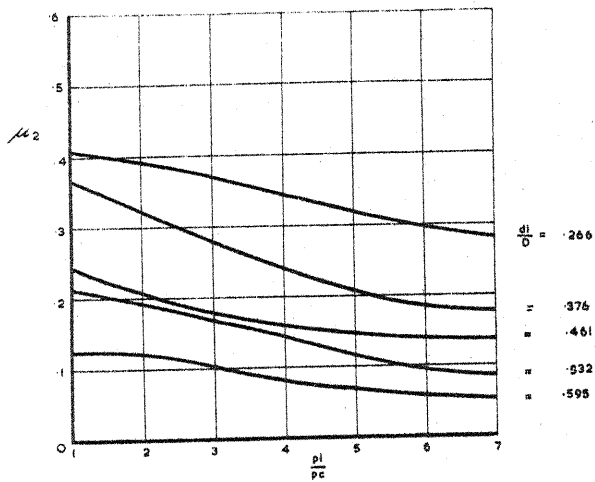


FIG. 60. VARIATION OF COLD MASS FLOW RATIO WITH PRESSURE RATIO AT OPTIMUM VALVE SETTING AND OPTIMUM COLD OUTLET DIAMETER.

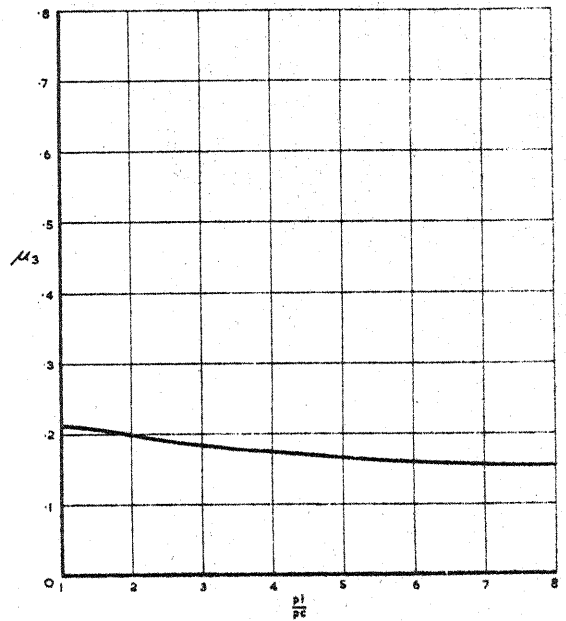


FIG. 61. VARIATION OF COLD MASS FLOW RATIO WITH PRESSURE RATIO FOR OPTIMUM VALVE SETTING, OPTIMUM COLD OUTLET DIAMETER RATIO AND OPTIMUM INLET DIAMETER.

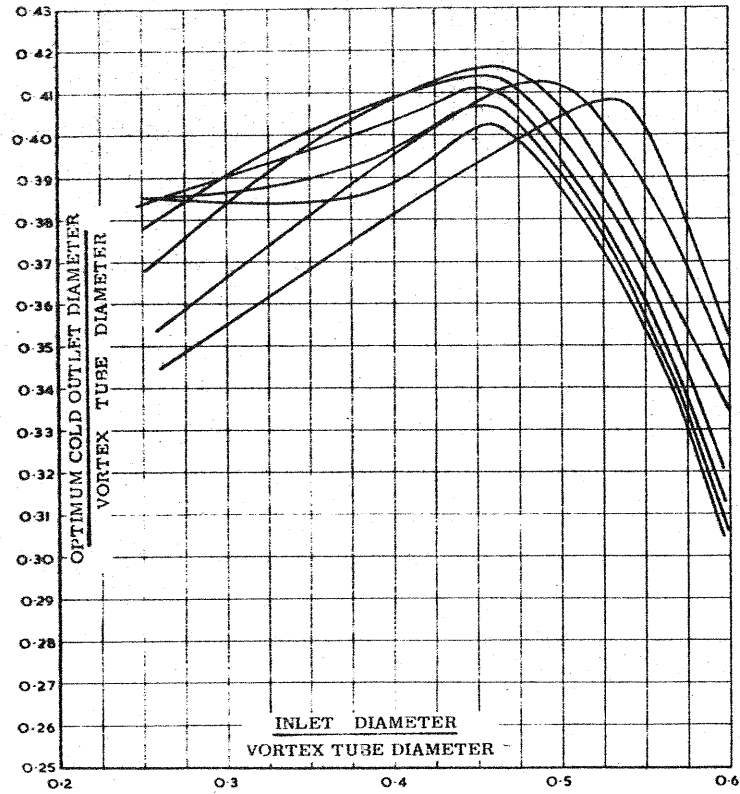


FIG. 62. VARIATION OF OPTIMUM COLD OUTLET DIAMETER WITH INLET DIAMETER

$$\frac{p_i}{p_c} = 1.5, 2, 3, 4, 5, 6 \text{ and } 7$$

$$\left(\frac{d_c}{D}\right)_1 \sim \frac{d_i}{D}$$

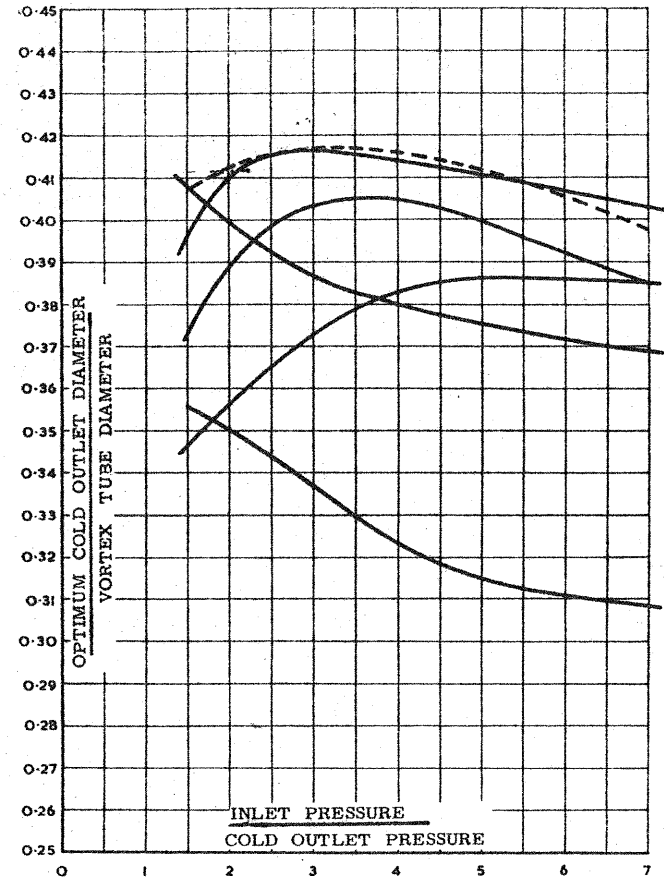


FIG. 63. VARIATION OF OPTIMUM COLD OUTLET DIAMETER WITH PRESSURE RATIO FOR FIXED INLET DIAMETERS,  $\frac{d_i}{D} = .266, .376, .461, .532 \text{ \& } .595$

$$\left(\frac{d_c}{D}\right)_1 \sim \frac{p_i}{p_c}$$

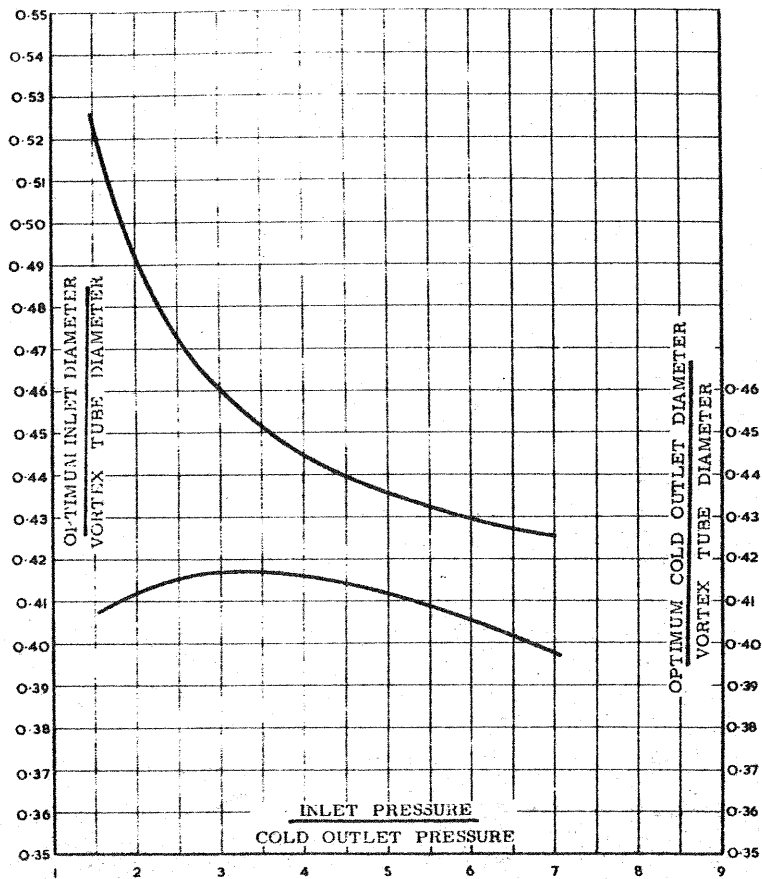


FIG. 64 OPTIMUM INLET AND COLD OUTLET DIAMETERS FOR MAXIMUM TEMPERATURE DROP

$$\left(\frac{d_1}{D_1}\right) \sim \frac{p_1}{p_c}$$

$$\left(\frac{d_c}{D_c}\right) \sim \frac{t_1}{c}$$

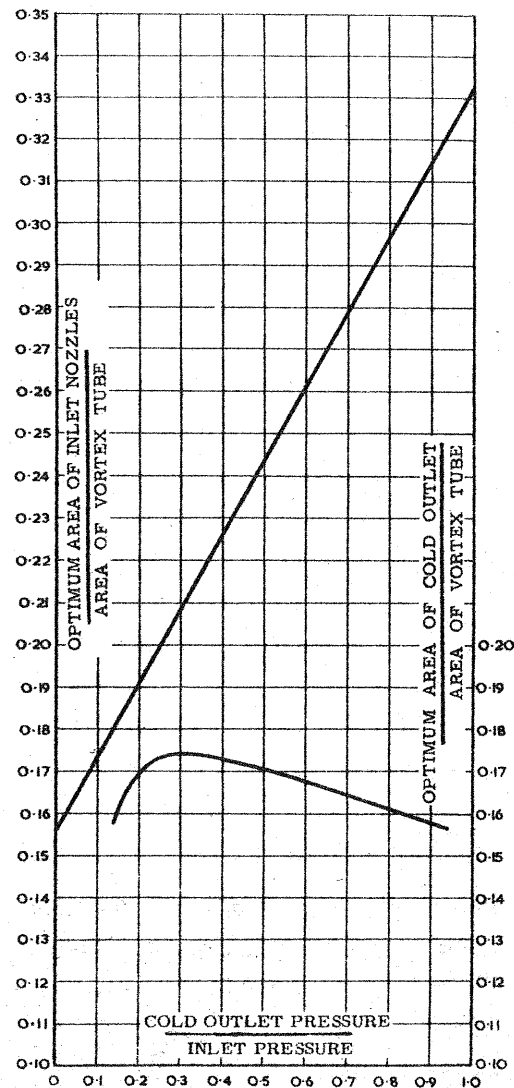


FIG. 65. OPTIMUM INLET AND COLD OUTLET AREAS FOR MAXIMUM TEMPERATURE DROP.

$$\left(\frac{A_1}{A}\right)_1 \sim \frac{p_1}{p_c}$$

$$\left(\frac{A_c}{A}\right)_2 \sim \frac{p_1}{p_c}$$